

THE INFLUENCE OF VARIABLE OPERATING CONDITIONS ON THE DESIGN AND EXPLOITATION OF FLY ASH PNEUMATIC TRANSPORT SYSTEMS IN THERMAL POWER PLANTS

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(Received: January 31, 2007 ; Accepted: April 10, 2008)

Abstract - The efficiency of an air-slide pneumatic conveying system depends, first of all, on several basic elements chosen or calculated during the design of a plant: air-slide design parameters, air mover characteristics, as well as the physical and chemical properties of the material to be transported. However, during the exploitation of this type of system which is used for handling ash in thermal-power plants, either gradual and/or sudden changes in the operating conditions can arise. This may be due to changes both in the proportion of ash content, and in the flow characteristics of the porous membrane. The consequences of changes in these conditions on the performance of the ash handling system are analyzed, based upon the results of the experimental work carried out on the test rig at the Faculty of Mechanical Engineering in Belgrade, and upon the on-site measurements at the thermal-power plant "Nikola Tesla B".

Keywords: Electrostatic precipitator; Pulverized fly ash; Air-slide conveying system.

INTRODUCTION

The investigation that is reported in this paper has been initiated by recurring problems with the pneumatic transport of pulverized fly ash in the air-slide system at the thermal-power plant "Nikola Tesla B" near Belgrade. The system works successfully most of the time, but clogging of the air-slide occurs every few months and urgent intervention is needed in order to prevent excessive filling of the precipitators and consequently plant shut-down.

ASH HANDLING SYSTEM IN THERMAL-POWER PLANT "NIKOLA TESLA B"

Low grade lignite, from the adjacent surface mine, is used as a fuel in the 1200 MW power plant "Nikola

Tesla B". This kind of lignite has a high and variable ash content. After being separated in the electrostatic precipitator, fly ash is transferred by several parallel air-slides (15.2 m long, 200 mm wide, 380 mm high, 3° angle) from the bottom of the precipitator hoppers to the ejector hydraulic conveying system, which takes it to the main slurry tank. The designed capacity of each air-slide is 15 t/h of ash. An ejector hydraulic system transports ash to the main slurry tank. Finally, the centrifugal slurry pumps are then used to transport ash to a disposal area, some 5 ÷ 8 km away from the plant. This air-slide system operates very efficiently with the ash obtained from standard quality coal. In actual fact, it seems that, for this kind of ash, the system is over-designed, as the air-ash mixture flows in the chamber of the air-slide in a mode which is closer to a low velocity pneumatic conveying system than to a sliding fluidized bed.

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However, occasionally deposition of the ash occurs at the bottom of the air-slide chamber, as the air flow rate from the centrifugal fan is not sufficient to fluidize the ash and move it down the slide. It was obvious that the quality of the ash changed considerably from the standard and, if this kind of ash filled the air-slide for a long time, the stationary layer reached the height of the chamber and transport stopped completely.

In order to determine all the necessary flow characteristics of the different ash samples as well as the characteristics of the other relevant parts of the air-slide system (porous membrane, air mover, air supply lines), a comprehensive test programme was carried out both on site and at the Faculty of Mechanical Engineering in Belgrade.

AIR-SLIDE DESIGN PARAMETERS

In order to specify the air requirement for the air-slide system, two major parameters have to be determined:

1. air velocity (w_a) which will fluidize the product sufficiently for sliding down,
2. corresponding pressure drops through the bed (Δp_b) and through the porous membrane (Δp_m).

Recommendations for the air velocity (w_a), which can be found in the relevant literature, are always related to the minimum fluidizing velocity (w_{mf}). However, the recommendations vary from 10% higher than w_{mf} (for free flowing products with a narrow particle size distribution) up to 10 times the minimum fluidizing velocity in other cases. The problems are enhanced by the fact that there is still no absolutely reliable formula for prediction of the minimum fluidizing velocity. Therefore, in order to obtain the exact values, both the minimum fluidizing velocity and the corresponding pressure drop for the ash samples were measured on the laboratory test rig. The results were then compared with both the relevant literature values and the values which were measured on the real plant.

Pressure Drop Across Fluidized Bed

There are many proposed expressions for pressure drop across the bed. Two of them are given here and are compared with the obtained experimental results. They were originally derived by Kozeny and Carman (1937), for a laminar flow regime, and Ergun (1952), for any flow regime, but other authors improved them later, for example, [Wen and Yu (1996)]:

$$\frac{\Delta p_b}{H} = 200 \frac{(1-\epsilon_o)^2}{\phi_s^2 \epsilon_o^3} \frac{\mu w_f}{d^2} \quad (1)$$

$$\frac{\Delta p_b}{H} = 200 \frac{(1-\epsilon_o)^2}{\phi_s^2 \epsilon_o^3} \frac{\mu w_f}{d^2} + 1,75 \frac{1-\epsilon_o}{\phi_s \epsilon_o^3} \frac{\rho_f w_f^2}{d} \quad (2)$$

These relations are often used as a starting point for the determination of the minimum fluidizing velocity. Some of them use various, mainly experimental, correction factors.

Minimum Fluidizing Velocity

Numerous formulae can be found for the determination of the minimum fluidizing velocity and it is not the intention of this paper to list them all. All of them are based on both experimental data with different products and on the assumption that, at the point of incipient fluidization, the pressure gradient is equal to the net gravity force of the bed per unit area:

$$\frac{\Delta p}{H} = (\rho_p - \rho_f) g (1 - \epsilon_{mf}) \quad (3)$$

Later on, experimental attempts to find the relationship between the voidage at minimum fluidizing conditions (ϵ_{mf}) and sphericity (Φ_s) were carried out Wen and Yu (1966), although this relationship is not likely to be universal, as some other material characteristics should be involved too. However, these experiments led to rather acceptable relationship between Reynolds and Archimedes numbers:

$$Re_{mf} = \frac{w_{mf} d \rho_f}{\mu} \quad (4)$$

$$Ar_b = \frac{g d^3 (\rho_p - \rho_f) \rho_f}{\mu^2} \quad (5)$$

Wen and Yu (1996) plotted a large number of published test data points on this kind of graph (Fig. 1), along with the line obtained by substituting two relations between ϵ_{mf} and Φ_s , which they had found, into Ergun's equation:

$$Re_{mf} = \sqrt{33.7^2 + 0.0408 Ar_b} - 33.7 \quad (6)$$

Actually, this line looks like the best fit line through the test data points, in which case no relationship between ϵ_{mf} and Φ_s was required. A similar type of relation has been proposed by Baeyens and Geldart (1973):

$$Ar_b = 1823 Re_{mf}^{1.07} + 21.7 Re_{mf}^2 \quad (7)$$

This relation is plotted with a dashed line in Fig. 1, along with the previous one (full line). The difference between them is relatively small: for $410 < Ar_b < 4.5 \cdot 10^6$ relation 7 gives lower values for Re_{mf} up to 15% than relation 6; for $Ar_b > 4.5 \cdot 10^6$ it gives higher values for Re_{mf} up to 5% and for $Ar_b < 410$ it gives higher values for Re_{mf} up to 40% (at the minimum values of Ar_b). However, relation 7 is also within the band of test data points.

Finally, in a similar way, Leva (1959) derived his formula for w_{mf} starting from Carman-Kozeny's formula and using his tests:

$$w_{mf} = 0.00923 \frac{d^{1.82}}{v^{0.88}} \left(\frac{\rho_p}{\rho_f} \right)^{0.94}, \frac{m}{s} \quad (8)$$

where: $d(m)$, $v(m^2/s)$, ρ_p and ρ_f (kg/m^3). Because this took into account only the viscous effect, it should be used for $Re_{mf} < 10$ without any corrections (for $Re_{mf} > 10$, the correction factor should be applied). This relation is also plotted in Fig. 1 (with circular dots).

Although the proposed correction factor was applied, the difference still exist for $Re_{mf} > 500$. The minimum and maximum values in the test data band will be discussed later, along with the test data for ash samples (NT-6 and NT-9).

Pressure Drop Across Porous Membrane

This pressure drop can be obtained only by experiment and, due to the laminar flow regime, it can be presented in the following way:

$$\Delta p_m = k_m w_f, \text{ kPa} \quad (9)$$

where: k_m (kPa/m/s) is the membrane resistance coefficient and w_f (m/s) is the superficial air velocity which approaches the membrane. However, during the normal operation of the air-slide, the membrane resistance coefficient will gradually increase due to the penetration of the fine particles. This effect will be discussed later.

LABORATORY TEST PROGRAMME

The test programme, carried out under laboratory conditions, was aimed primarily at determining the flow characteristics of the ash during fluidization, as they are of vital importance for the design of the air-

slide parameters. Naturally, some other parameters were measured also (particle size and density, bulk density) in order to compare the obtained data with the ones proposed in the literature. Special attention has been paid to the determination of the resistance coefficients of different samples of porous membranes, used at the plant.

Ash Samples Characteristics

As each air-slide was fed from four points (different rows of the electrostatic precipitator), eight samples of standard quality ash were taken for further analysis (NT-1 to NT-8). Later, when deposition of the ash in the air-slide had been noticed, a sample of that ash was also taken for further analysis (NT-10). Another sample was formed in the laboratory, when sample NT-10 was separated in a fluidizing column into two fractions: top one (lighter and finer) and bottom one (heavier and coarser). The bottom one was tested too (NT-9) as it simulates the conditions which would appear in the air-slide at the point of clogging.

The value of particle density for the standard ash samples varied between 1400 and 1500 kg/m^3 . However, its value for sample NT-10 was higher (1680 kg/m^3) and for NT-9 it was significantly higher - 2170 kg/m^3 . The reason for this increase was higher SiO_2 content in samples NT-9 and NT-10, confirmed by chemical analysis.

The value of poured bulk density for the standard ash samples varied between 610 and 785 kg/m^3 . The variation in the value of poured bulk density ($\pm 12\%$) of these samples was expected, due to the different points from which the samples were taken. The poured bulk density for sample NT-10 was inside this range (715 kg/m^3), however, its value for sample NT-9 was significantly higher - 1180 kg/m^3 . The tapped bulk density values for all samples were 10 - 15% higher than the corresponding poured bulk density values, except for sample NT-9, which was only 6% higher (Table 1).

The mean particle diameter for standard quality samples was around $d = (80 - 120) \mu m$, however it was higher for both samples NT-9 and NT-10 (200 - 350 μm). As detailed results for only samples NT-6 (one of standard quality) and NT-9 (with higher SiO_2 content) are presented in this paper, Table 2 gives details of all mentioned characteristics of those two samples. The terminal velocity values have been calculated in the standard way, using the values from Table 2 and the well known experimental relationship between the drag coefficient C_D and the particle Reynolds number ($Re_t = v_t \cdot d/v$).

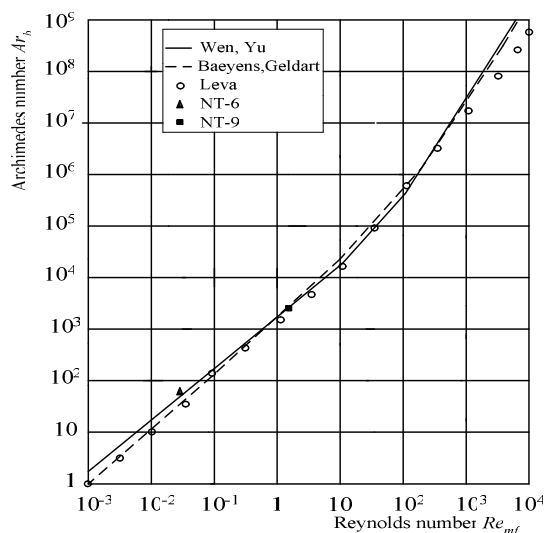


Figure 1: Comparison of some relationships for minimum fluidizing velocity

Table 1: Bulk density values for all samples

Sample	Poured bulk density ρ_{b0} (kg/m ³)	Tapped bulk density *		$\frac{\rho_{b60} - \rho_{b30}}{\rho_{b0}} \cdot 100$ (%)
		ρ_{b30} (kg/m ³)	ρ_{b60} (kg/m ³)	
NT-1	610	670	675	11
NT-2	685	765	780	14
NT-3	655	725	745	14
NT-4	720	780	790	10
NT-5	695	755	790	14
NT-6	740	800	820	11
NT-7	785	865	875	11
NT-8	725	825	835	15
NT-9	1180	1255	1255	6
NT-10	715	780	790	10

* ρ_{b30} - tapped bulk density 30 s after of fluidization,

* ρ_{b60} - tapped bulk density 60 s after of fluidization

Table 2: Characteristics of ash samples: NT-6 and NT-9

Sample	Bulk Density		Particle Density	Voidage	Mean Particle Diameter	Terminal Velocity
	poured	tapped				
	ρ_{b0} (kg/m ³)	ρ_{b60} (kg/m ³)	ρ_p (kg/m ³)	-	d (μ m)	v_t (m/s)
NT-6	740	820	1450	0.490	105	0.48
NT-9	1180	1255	2170	0.456	320	2.25

Fluidization Test Results

Fluidization tests were carried out in a standard type of experimental rig (Fig. 2) with the column diameter $D = 150$ mm. As very low air flow rates were used, a combination of three metering devices was used to accurately cover the whole range (orifice plate, rotameter and Venturi's nozzle). The data were recorded as the fluidizing air velocity was both

increasing and decreasing. Different initial bed heights were used, ranging from 70 to 300 mm.

The test data for the pressure gradient and the superficial fluidizing air velocity for two ash samples (NT-6 and NT-9) are plotted in Fig. 3. In order to compare these pressure gradient data with equations 1 and 2, the sphericity Φ_s was calculated, using Wen and Yu (1966). The obtained values were $\Phi_s = 0.628$ (for NT-6) and $\Phi_s = 0.722$ (for NT-9) and these

results are then plotted in Fig. 4 (for NT-6) and in Fig. 5 (for NT-9).

A remarkable agreement can be seen with all test results for sample NT-9 and for the lower air velocities for sample NT-6. A significant discrepancy appears only for sample NT-6 at higher air velocity, above the experimental value of the minimum fluidizing velocity. Actually, the obtained constant value of pressure gradient (4.7 kPa/m) was significantly lower than the expected one from equation 3 (7.3 kPa/m). The results for this pressure gradient with other samples, which are not presented here, were similar:

$$\Delta P_b / H = (4.2 - 4.5) \text{ kPa/m for samples with } \rho_{b0} = (610 - 700) \text{ kg/m}^3,$$

$$\Delta P_b / H = (4.7 - 5.2) \text{ kPa/m for samples with } \rho_{b0} = (715 - 785) \text{ kg/m}^3.$$

This discrepancy occurred partly due to the presence of uncombusted coal particles with very rough surfaces, which increased voidage locally, to a certain extent due to channeling, although it was not very remarkable.

Both the experimental values for the minimum fluidizing velocity w_{mf} as well as some of the calculated ones can be obtained from Fig. 4 and 5. These values, together with the other calculated ones, are given in Table 3. The values for w_{mf} for

other ash samples (with slightly different physical characteristics) were in the range (0.0035-0.0065) m/s.

If one wants to determine the value for w_{mf} from Fig. 1, using value Ar_b only, the range would be $w_{mf} = (0.0036 - 0.0086) \text{ m/s}$ for NT-6 and $w_{mf} = (0.0425 - 0.1180) \text{ m/s}$ for NT-9. It is now obvious that all values from Table 3 fall in this range. However, according to Wen and Yu (1966), the ratio between maximum and minimum values of Re_{mf} and hence w_{mf} , for $Ar_b < 10^5$ is in the order of 2.5! Therefore, the prediction of w_{mf} , relying on any relation $Re_{mf} = f(Ar_b)$ can be very inaccurate. Some more material characteristics need to be involved too.

Resistance of the Porous Membrane

The porous membrane at the "Nikola Tesla B" power plant is made of compacted wool, around 30mm thick. As its membrane resistance coefficient k_m increases during normal operation of the ash handling plant, the point is reached when the fan can no longer serve the duty. Therefore, four different samples of the membranes were tested in the laboratory in order to determine their lifetime. The obtained test results are shown in Fig. 6.

The values of the coefficient k_m , as well as other characteristics of the membrane samples are given in Table 4. It can be seen that the coefficient k_m depends strongly on the exploitation time and this has to be considered in the design.

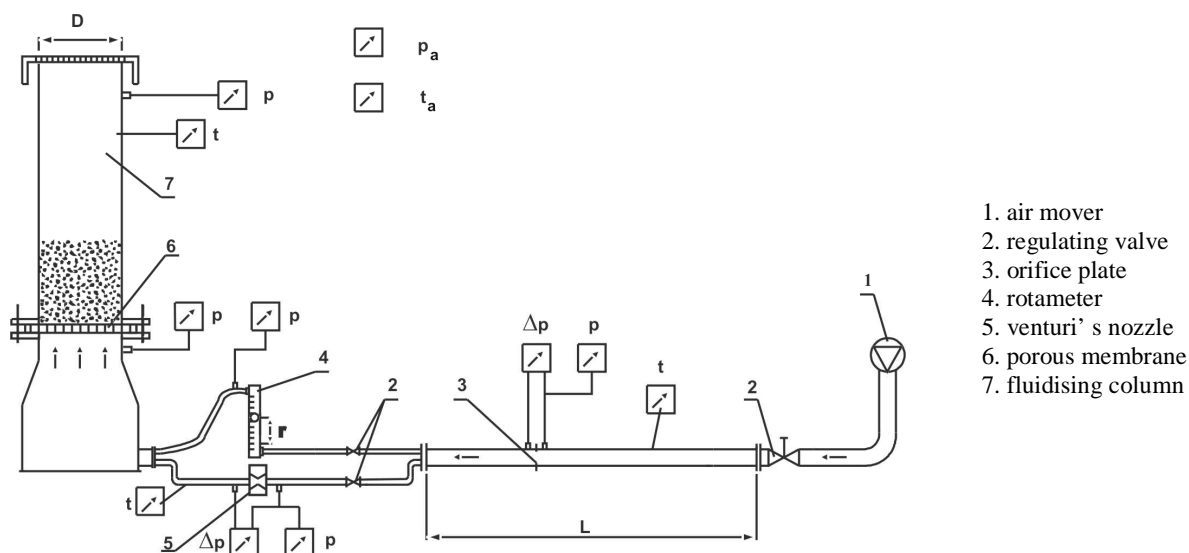


Figure 2: Experimental apparatus for determination of fluidizing properties

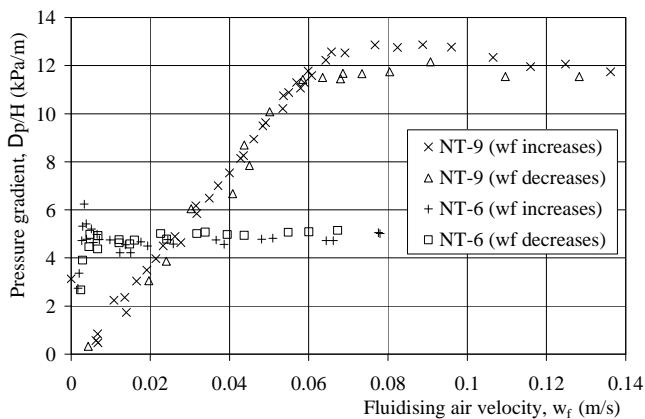


Figure 3: Pressure gradient test data for ash samples

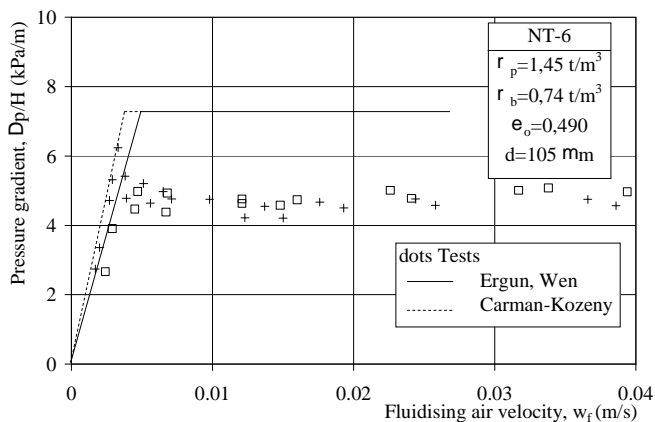


Figure 4: Comparison of test data for NT-6 with some proposed formulae

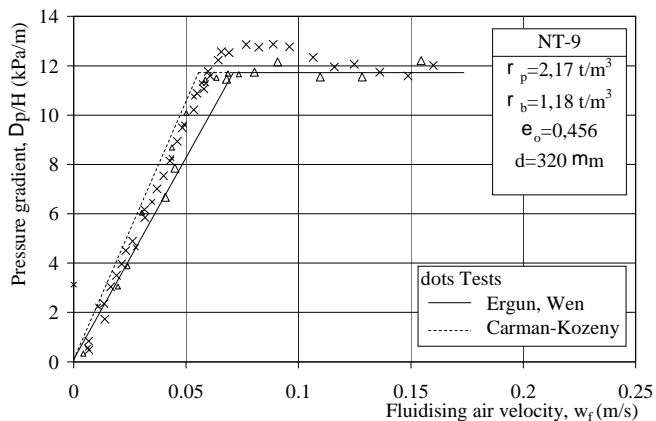


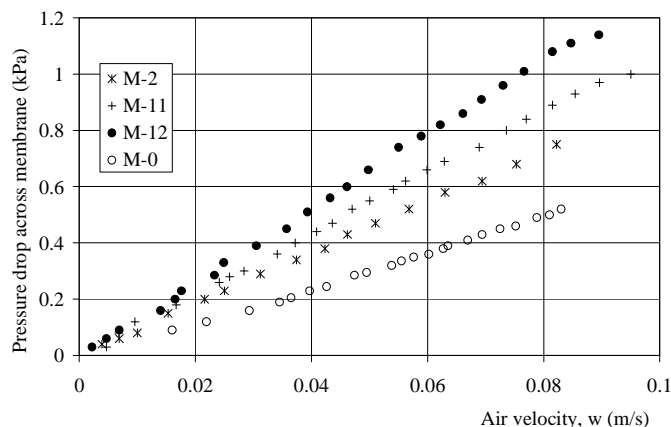
Figure 5: Comparison of test data for NT-9 with some proposed formulae

Table 3: Values of minimum fluidizing velocity w_{mf} obtained by different sources

Source	NT-6 ($Ar_b = 60$)			NT-9 ($Ar_b = 2541$)		
	Re_{mf}	w_{mf}	Error	Re_{mf}	w_{mf}	Error
	-	m/s	%	-	m/s	%
Test	0.026	0.0038	-	1.38	0.0650	-
Wen, Yu	0.036	0.0052	+37	1.51	0.0711	+9
Baeyens, Geldart	0.041	0.0059	+55	1.34	0.0634	-2
Carman-Kozeny	0.027	0.0039	+3	1.14	0.0539	-17
Leva	0.051	0.0073	+92	1.72	0.0811	+25

Table 4: Characteristics of the porous membrane

Membrane Sample	Thickness (mm)	Used for	Coefficient k_m (kPa/m/s)
M-0	30	New	5.95
M-2	30	8 months	8.82
M-11	25	2 years	10.72
M-12	30	2 years	12.92

**Figure 6: Test data for porous membrane**

IMPLICATION OF THE RESULTS OF AIR SLIDE DESIGN PARAMETERS

In order to analyze the air slide operating conditions, the total pressure drop $\Delta p_t = \Delta P_m + \Delta P_b$ was measured in the fluidizing rig using the same initial height of ash ($H = 100\text{mm}$) as estimated in the air slide and new porous membrane. The results are presented in Fig. 7.

Another set of tests was carried out at the plant where fan characteristics were measured. It showed that superficial air velocity through the bed was $w_a = 0.051\text{ m/s}$, hence $8 \div 15$ times higher than w_{mf} for standard quality ash, which is more than sufficient for ash handling in an air-slide system. In actual fact, another important parameter in the design of the air-slide is often forgotten: the ratio of vertical to horizontal cross-sectional areas of the air-slide (or length to height ratio). This ratio determines the superficial axial air velocity distribution along the air-slide, once the fluidizing air velocity is set. The ratio at the "Nikola Tesla B" plant was 40, which means that the axial air velocity reaches $w_{as} = 2.04\text{ m/s}$ at the end, and confirms the visual conclusion on the low velocity pneumatic conveying mode of flow in the air-slide.

On the other hand, velocity w_a is less than w_{mf} required for the ash with high SiO_2 content, which occurs from time to time, and is not sufficient to fluidize it. Therefore, if only a 20% higher value of minimum fluidizing velocity is adopted ($w_{al} = 0.078\text{ m/s}$), it will give very high velocity along the air slide ($w_{as} = 3.12\text{ m/s}$).

Another consequence of a high fluidizing air velocity is the entrainment of fine particles in the air

stream leaving the surface of the bed. Theoretically, this happens when the upward velocity exceeds the particle terminal velocity. Fig. 8 shows the theoretical relationship between terminal velocity and particle size for the standard quality ash. During the fluidizing laboratory tests, it was noticed that a considerable amount of ash was leaving the bed at the velocity ($0.060 - 0.070$) m/s, which corresponds to particle size ($37 - 42$) μm . At the velocity $w_a = 0.051\text{ m/s}$, the maximum particle size which leaves the bed is still high ($34\text{ }\mu\text{m}$).

There are several ways in which to meet these wide range air requirements, as a single machine, normally a centrifugal fan, cannot meet the whole range and should be designed to handle standard quality ash; in the observed case the required value of fluidizing air velocity was determined as $w_a = (0.02 \div 0.03)\text{ m/s}$, which is half of the present value. Using an identical stand-by air mover, it would only be possible to double the flow rate, which still would not be sufficient for the worst case (sample NT-9). Therefore, a larger size air mover could be used as a stand-by machine, but it might not be acceptable for various reasons. Another possibility could be implemented if one machine supplied air to several air-slides (at "Nikola Tesla B" plant four air-slides). Regulating and control systems should then be incorporated which would enable the full air mover flow rate to be redistributed to a single air-slide. In the observed case, the flow rate to a single air-slide could be increased four times with one mover and eight times with two of them.

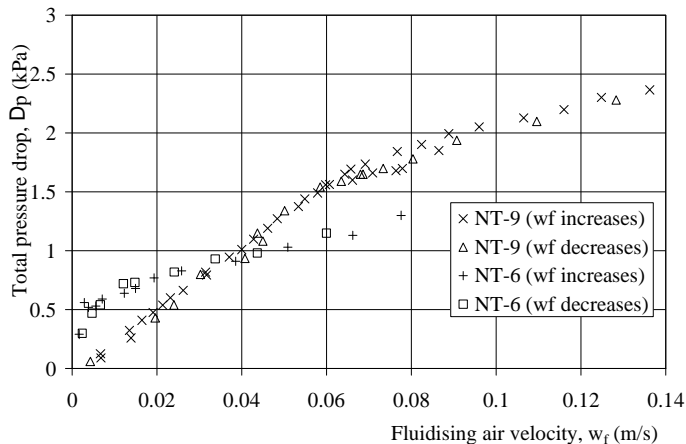


Figure 7: Total pressure drop data for NT-6 and NT-9

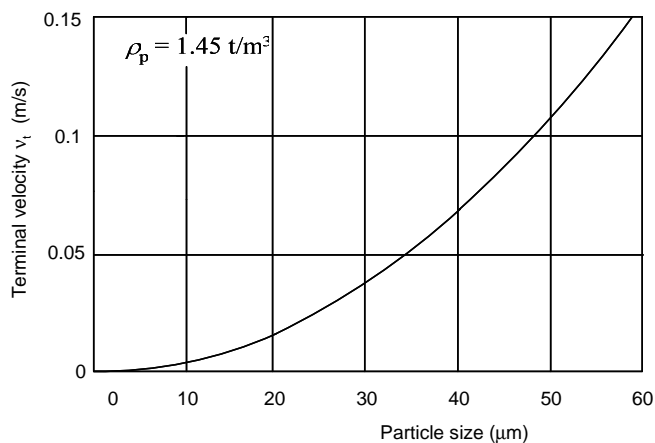


Figure 8: Terminal velocity for standard quality ash

CONCLUSION

Investigation has shown that the pressure signal from the air-slide will not necessarily indicate that the ash quality changed, as total pressure drop remains almost the same at air velocity $0.03 \div 0.04$ m/s. At lower values standard quality ash gives higher resistance, being less permeable, whereas at higher values it gives lower pressure drop. Thus, if the total pressure drop data analysis for a designed air velocity shows that there is no significant difference, then either chemical analysis of the coal or visual control should be used as a signal for implementing emergency measures.

Total pressure drop Δp_t , which the fan should deliver will change significantly as the porous membrane gets filthy. At the air flow rate measured at the plant and standard quality ash, Δp_t would be 1.0 kPa with a new membrane and 1.4 kPa with a membrane used for two years. In this case the total pressure drop can be used as a reliable indicator for signaling the need for change a porous membrane.

This analysis deals with various aspects of the design of reliable flow in an air-slide system. The prediction of some flow parameters (minimum fluidizing velocity, pressure drop across fluidized bed) offered in the literature can be used as a guideline. However, in order to design key parameters, required fluidizing velocity and product flow rate, reliable experimental data is essential. Furthermore, the change in flow characteristics of some elements of an air-slide system (product, porous membrane, air mover) should be considered at the initial stage of the design in order to prevent operating problems.

NOMENCLATURE

Δp_b	pressure drop through the bed	kPa
Δp_m	pressure drop through the porous membrane	kPa
Δp_t	total pressure drop	kPa
k_m	membrane resistance coefficient	kPa/m/s

H	bed height	m	Φ_s	Sphericity	(-)
ϵ_o	porosity of fixed bed	(-)			
ϵ_{mf}	minimum fluidization porosity	(-)			
w_a	superficial air velocity through the bed	m/s			
w_{mf}	minimum fluidizing air velocity	m/s			
w_f	fluidizing air velocity	m/s			
v_t	terminal velocity	m/s			
ρ_f	air density	kg/m ³			
μ	dynamic viscosity of air	kg/m-s			
ν	kinematic viscosity of air	m ² /s			
d	mean particle diameter	μm			
ρ_p	particle density	kg/m ³			
ρ_b	bulk density	kg/m ³			
Ar_b	Archimedes number	(-)			
Re_{mf}	Reynolds number	(-)			
C_D	drag coefficient	(-)			

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