# Deformation and Stress Fields in Thin Metallic Partially Fixed Plate Induced by Electromagnet with Constant Flux Obtained by Analytical Method and FEM 

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#### Abstract

In the paper the behavior of thin elastic metallic partially fixed plate produced by electromagnet with constant flux is considered. Plate is simply supported along three edges and fixed along the fourth one. As nonstationary electromagnetic field in one part of the plate varies with time, conducting currents appear. Distribution of the power of eddy-current losses is obtained by using complex calculation. The power of the volume heat source (Joule's heat) depends of permeability and conductivity of the plate and frequency of the alternative-current in electromagnet. Characteristics of the electromagnet are described by using Pulsation, Heaviside and similar functions. Differential equation governing temperature field is solved using by integral transform technique. The mathematical form of the transversal vibrations is obtained in analytical way using only simple finite Fourier transforms because the differential equation describing transverse vibrations is adapted to form which enabling very easy simulation of the bending moments along the fixed edges. Vibrations and stress in the middle surface of the plate are calculated by using program package KOMIPS, based on the finite element method.


Key words: electromagnet, temperature field, stress, deformation, finite element method

## 1. Introduction

Vibrations of the thin metallic plates from soft ferromagnetic materials are described using four coupled systems of differential equations. The first system is a system of Maxwell's equations (with the relations for slowly moving media and modified Ohm's low) [5]:

$$
\begin{equation*}
\operatorname{rot} \mathbf{H}=\mathbf{J}+\frac{\partial \mathbf{D}}{\partial \mathrm{t}}, \quad \operatorname{rot} \mathbf{K}=-\frac{\partial \mathbf{B}}{\partial \mathrm{t}}, \quad \operatorname{div} \mathbf{D}=0, \quad \operatorname{div} \mathbf{B}=0 . \tag{1.1}
\end{equation*}
$$

where the following notation is applied: $\mathbf{H}$-magnetic intensity vector, $\mathbf{K}$-electric intensity vector, $\mathbf{B}$-vector of magnetic induction, D- vector of electric induction and J-current density vector.

Temperature field in thin plates can be described using two values: $\tau_{0}$ - temperature in the middle surface and $\tau_{1}$ - the rate of temperature across the plate thickness [7]. So, second system of equations is consisted of two partial differential equations (1.2), where $\kappa$ is coefficient of thermal intensity, $\eta^{*}$ is representing the coupling between the temperature and the deformation fields, $\varepsilon^{\prime}$ is deformation in the middle surface of the plate, $h$ is the plate thickness, $\lambda_{0}$ is heat conduction coefficient, $\sigma$ is electric conductivity and $\nabla_{1}^{2}$ is Laplace operator [2]:

$$
\begin{gather*}
\left(\nabla_{1}^{2}-\beta_{k}-\frac{1}{\kappa} \frac{\partial}{\partial t}\right) \tau_{k}+\frac{\beta_{k}^{k}}{h}\left[x_{3}^{k} \frac{\partial \theta}{\partial x_{3}}\right]_{-\frac{h}{2}}^{\frac{h}{2}}-\eta^{*} \varepsilon_{k}=-\frac{\beta_{k}^{k} \int_{-\frac{h}{2}}^{\frac{h}{2}} W\left(x_{1}, x_{2}, x_{3}, t\right) x_{3}^{k} d x_{3}}{h \lambda_{0}}, \\
\beta_{k}=\left\{\begin{array}{ll}
0, & k=0 \\
\frac{12}{h^{3}}, & k=1
\end{array}, \quad \varepsilon_{k}=\left\{\begin{array}{cc}
\varepsilon^{\prime}, & k=0 \\
\nabla_{1}^{2}\left(\partial_{t}^{2} w\right), & k=1
\end{array}, \quad W=W_{S}+W_{H}+\frac{J^{2}}{\sigma}, \quad(k=0,1) .\right.\right. \tag{1.2}
\end{gather*}
$$

Losses in a plate $W\left(x_{1}, x_{2}, x_{3}, t\right)$ are consisted of three factors: volume heat source intensity, hysterisis losses and Joule's heat (eddy-current losses).

In the consideration of the vibrations of the plate, the assumption that the longitudinal vibrations are independent of the transverse vibrations is taken. Transverse vibrations can be obtained using by next differential equation [2]

$$
\begin{gather*}
D \nabla_{1}^{4} w+D(1+v) \alpha_{t} \nabla_{1}^{2} \tau_{1}+\rho h\left(\partial_{t}^{2} w\right)-\frac{\rho h^{3}}{12} \nabla_{1}^{2}\left(\partial_{t}^{2} w\right)=\left(\sigma_{33}^{+}-\sigma_{33}^{-}\right)+\left(T_{33}^{+}-T_{33}^{-}\right)+  \tag{1.3}\\
\quad+\frac{h}{2} \frac{\partial}{\partial x_{i}}\left(\sigma_{i 3}^{+}+\sigma_{i 3}^{-}\right)+\frac{h}{2} \frac{\partial}{\partial x_{i}}\left(T_{i 3}^{+}+T_{i 3}^{-}\right)+\int_{-\frac{h}{2}}^{\frac{h}{2}}\left(X_{i, i}+f_{i, i}\right) x_{3} d x_{3}+\int_{-\frac{h}{2}}^{\frac{h}{2}} X_{3} d x_{3},
\end{gather*}
$$

where $w$ denotes deflection of the plate in $X_{3}$-direction, $v$ is Poisson ratio, $\alpha_{t}$ is coefficient of thermal expansion, $D$ is flexural rigidity of the plate, $E$ is modulus of elasticity, $X_{i}$ and $f_{i}$ are the components of mechanical force and Lorenz force, $\sigma_{i j}$ and $T_{i j}$ are mechanical and magnetic stress tensors ( + on the upper and - on the lower side of the plate) and $\rho$ is plate density.

Vibrations in the middle surface of the plate are defined by the equation

$$
\begin{gather*}
\Delta_{2}^{2}\left(\Delta_{1}^{2} F+E h \alpha_{t} \tau_{0}\right)=0,  \tag{1.4}\\
\Delta_{i}^{2} \equiv \nabla_{1}^{2}-\frac{1}{C_{i}^{2}} \partial_{t}^{2} \quad(i=1,2), \quad C_{1}^{2}=\frac{D_{z}}{\rho h}, \quad C_{2}^{2}=\frac{E}{2 \rho(1+v)},
\end{gather*}
$$

where F is Airy-s Stress function. Forces $N_{i j}$ can be expressed as [7]

$$
N_{i j}=-F_{, i j}+\delta_{i j}\left(\nabla_{1}^{2}-\frac{1}{2 C_{2}^{2}} \partial_{t}^{2}\right) F \quad(i, j=1,2)
$$

Presented system of equations has to be added with an appropriate boundary and initial conditions.

## 2. Electromagnet with constant flux

As a result of time-changing magnetic induction, in one part of the plate ( $d \times c \times h$ ) conducting currents appear. The calculation of the Joule's heat intensity is done by using local coordinate system ( $x, y, z$ ) with next approximations (Fig. 2.1): component of the magnetic induction $B_{y}$ is zero; component $B_{z}$ is minor compared to the component $B_{x}$. So, $\mathrm{H}_{\mathrm{x}}$ and $\mathrm{K}_{\mathrm{y}}$ components can describe electromagnetic field in the plate, which are simplified Maxwell's equations (1.1). As the changes in $x$ and $y$ directions are minor, differential equation is [6]

$$
\begin{equation*}
\frac{\partial^{2} \underline{\underline{H}}_{x}}{\partial z^{2}}-\gamma \frac{\partial \underline{H}_{x}}{\partial t}=0, \quad \gamma^{2}=j \sigma \mu \omega, \quad \gamma=(1+j) \sqrt{\frac{\sigma \mu \omega}{2}}=(1+j) k, \tag{2.1}
\end{equation*}
$$

where $\mu$ is magnetic permeability of the plate material and $\omega$ is angular frequency of the magnetic field.


Fig. 2.1 Electromagnet with constant flux

The solution can be represented in the form $\underline{H}_{x}=\underline{C}_{1} e^{\gamma /}+\underline{C}_{2} e^{-\lambda /}$, where coefficients $\underline{C}_{1}, \underline{C}_{2}$ can be obtained from the boundary conditions $\left(\underline{H}_{0}=\underline{H}_{x}(z=0)\right)$ and they are

$$
\begin{equation*}
\underline{C}_{1}=-\frac{H_{0} e^{-\varkappa h}}{2 \operatorname{Sh} \gamma h}, \quad \underline{C}_{2}=\frac{H_{0} e^{\jmath h}}{2 \operatorname{Sh} \gamma h} . \tag{2.2}
\end{equation*}
$$

Conducted currents and Joule's heat are defined by [2]

$$
\begin{gather*}
\underline{J}_{x}=\frac{\partial \underline{H}_{x}}{\partial z}=-\underline{H}_{0} \gamma \frac{\operatorname{Ch} \gamma(z-h)}{S h \gamma h},  \tag{2.3}\\
W=\frac{1}{2} \frac{\left|\underline{J}^{2}\right|}{\sigma}=\frac{k^{2} H_{0}^{2}}{\sigma} \frac{\operatorname{Ch} 2 k(z-h)+\cos 2 k(z-h)}{C h 2 k h-\cos 2 k h} . \tag{2.4}
\end{gather*}
$$

## 3. Temperature field

Let the initial and the boundary conditions are

$$
\begin{equation*}
\left.\theta\right|_{t=0}=0,\left.\quad \theta\right|_{x_{1}=0, a}=0,\left.\quad \theta\right|_{x_{2}=0, b}=0,\left.\quad \frac{\partial \theta}{\partial x_{3}}\right|_{x_{3}= \pm \frac{h}{2}}=0 . \tag{3.1}
\end{equation*}
$$

Appropriate values of the power of the heat source from equation (1.2) are

$$
\begin{align*}
W_{0}^{*} & =\int_{0}^{h} W(z) d z=\frac{k H_{0}^{2}}{2 \sigma} \frac{S h 2 k h+\sin 2 k h}{C h 2 k h-\cos 2 k h}  \tag{3.2a}\\
W_{1}^{*} & =\int_{0}^{h} W(z)\left(\frac{h}{2}-z\right) d z=\frac{h k H_{0}^{2}}{4 \sigma}\left(\frac{\operatorname{Sh} 2 k h+\sin 2 k h}{C h 2 k h-\cos 2 k h}-\frac{1}{k h}\right) \tag{3.2b}
\end{align*}
$$

The position of electromagnet is presented on figure 3.1 and described by the relation $\Pi\left(\frac{x_{1}-e}{c}\right) \Pi\left(\frac{x_{2}-f}{d}\right)$.


Fig. 3.1 Position of electromagnet
Fig 3.2 Position of mechanical forces
Differential equations describing temperature field have the next form

$$
\left(\nabla_{1}^{2}-\beta_{k}-\frac{1}{\kappa} \partial_{t}\right) \tau_{k}=-\frac{\beta_{k}^{k}}{\lambda_{0} h} W_{k} \Pi\left(\frac{x_{1}-e}{c}\right) \Pi\left(\frac{x_{2}-f}{d}\right) H(t), \quad \beta_{k}=\left\{\begin{array}{ll}
0, & k=0  \tag{3.3}\\
\frac{12}{h^{3}}, & k=1
\end{array}, \quad(k=0,1)\right.
$$

Subjected to the boundary conditions (3.1) the equation (3.3) can be solved by using the integral-transform technique. Applying double Fourier finite-sine transform and Laplace transform we arrive to the next solution

$$
\begin{equation*}
\tau_{k}=\frac{4}{a b} \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} \frac{4 \beta_{k}^{k} W_{k}}{\lambda_{0} h} \frac{\sin \alpha_{n} e \sin \alpha_{m} f}{\beta_{k}+\Delta_{m n}} \frac{\sin \alpha_{n} \frac{c}{2}}{\alpha_{n}} \frac{\sin \alpha_{m} \frac{d}{2}}{\alpha_{m}}\left[1-e^{-\kappa\left(\beta_{k}+\Delta_{m n}\right) t}\right] H(t) \sin \alpha_{n} x_{1} \sin \alpha_{m} x_{2} \tag{3.4}
\end{equation*}
$$

$$
\Delta_{m n}=\alpha_{n}^{2}+\alpha_{m}^{2}=\left(\frac{n \pi}{a}\right)^{2}+\left(\frac{m \pi}{b}\right)^{2}
$$

Numerical example is given for the steel rectangular plate dimension $45 \times 24 \times 0.1 \mathrm{~cm}$; position of electromagnet is defined with parameters: $\mathrm{e}=15 \mathrm{~cm}, \mathrm{f}=7.5 \mathrm{~cm}, \mathrm{c}=6 \mathrm{~cm}, \mathrm{~d}=9 \mathrm{~cm}, \mathrm{~S}=18 \mathrm{~cm}^{2}$ (Figure 4.1). Material constants are: $\lambda_{0}=0.5 \mathrm{~W} / \mathrm{cmK}, \sigma=10^{7} \mathrm{~S} / \mathrm{m}$ and $\mu_{\mathrm{r}}=1000$. Electromagnetic field parameters in the plate are: $\mathrm{H}_{0}=2000 \mathrm{~A} / \mathrm{m}\left(\mathrm{B}_{0}=2.5 \mathrm{~T}\right)$ and $\mathrm{f}=100 \mathrm{~Hz}$.

On diagram in Figure 3.3 the maximal value of the temperature $\tau_{0}$ in the middle surface of the plate is presented as a function of time $t$. Figure 3.4 shows distribution of the isotherm lines in the middle surface for the stationary state.


Fig. 3.3. Maximal temperature in the plate $\tau_{0}$ as a function of time $t$


Fig. 3.4. Isotherm lines in the middle surface of the plate, $\tau_{0}=0 \div 10\left(\right.$ step $\left.1^{\circ} \mathrm{C}\right)$

## 4. Vibrations. Stress field

### 4.1 Transversal vibrations

The solution of the transversal vibrations is founded in analytical form for the plate simply supported along three edges ( $x_{1}=a, x_{2}=0, b$ ) and fixed along the fourth one ( $x_{1}=0$ ) (Fig. 4.1). Boundary conditions have the form [4]

$$
\begin{gather*}
\left.w\right|_{x_{1}=0, a}=0,\left.\frac{\partial w}{\partial x_{1}}\right|_{x_{1}=0}=0,\left.M_{11}\right|_{X_{1}=a}=\left.\left[\frac{\partial^{2} w}{\partial x_{1}^{2}}+v \frac{\partial^{2} w}{\partial x_{2}^{2}}+(1+v) \alpha_{t} \tau_{1}\right] D\right|_{X_{1}=a}=0, \\
\left.w\right|_{X_{2}=0, b}=0,\left.M_{22}\right|_{X_{2}=0, b}=\left.\left[v \frac{\partial^{2} w}{\partial x_{1}^{2}}+\frac{\partial^{2} w}{\partial x_{2}^{2}}+(1+v) \alpha_{t} \tau_{1}\right] D\right|_{X_{2}=0, b}=0 . \tag{4.1}
\end{gather*}
$$



Initial conditions are responsible to the natural undeformed state

$$
\begin{equation*}
\left.w\right|_{t=0}=0,\left.\quad \frac{\partial w}{\partial t}\right|_{t=0}=0 \tag{4.2}
\end{equation*}
$$

Fig. 4.1 Boundary conditions

The position of the appropriate mechanical forces is shown on figure 3.2. If $B_{v}$ and $H_{v}$ are magnetic induction and magnetic field in air gaps, $p_{0}$ Maxwell stress and $S$ surface of one air gap, mechanical forces can be represented (using Impulsive Dirac functions $\delta$ ) by the following relations

$$
\begin{equation*}
p_{0}=\frac{1}{2} H_{0} B_{0}, \quad p_{0} S \delta\left(x_{1}-e\right)\left[\delta\left(x_{2}-g\right)+\delta\left(x_{2}-q\right)\right] \tag{4.3}
\end{equation*}
$$

Using differential equation (1.3) we can form the appropriate equation for the stationary problem $(t \rightarrow \infty)$ by simulated moment along the edge $x_{1}=0$ throw the stress $\sigma_{13}$ on the next way

$$
\begin{equation*}
\nabla_{1}^{4} w+(1+v) \alpha_{t} \nabla_{1}^{2} \tau_{1}=\frac{h}{D} \frac{\partial}{\partial x_{1}}\left[\sigma_{13}\left(x_{2}\right) \delta\left(x_{1}\right)\right]+\frac{p_{0} S}{D} \delta\left(x_{1}-e\right)\left[\delta\left(x_{2}-g\right)+\delta\left(x_{2}-q\right)\right] \tag{4.4}
\end{equation*}
$$

By using relation (3.4), double Fourier finite-sine transform and the relation for the first derivative of $\delta$ function

$$
\int_{0}^{a} \frac{\partial \delta\left(x_{1}\right)}{\partial x_{1}} \sin \left(\alpha_{n} x_{1}\right) d x_{1}=-\alpha_{n}
$$

the solution for transversal vibrations can be represented in the form

$$
\begin{align*}
& w\left(x_{1}, x_{2}, t\right)=\frac{4}{a b} \sum_{m=1,3, \ldots n=1}^{\infty} \sum_{n m}^{\infty} w_{n m} \sin \alpha_{n} x_{1} \sin \alpha_{m} x_{2}  \tag{4.5}\\
& w_{n m}=\frac{\alpha_{t}(1+v) \tau_{1 n m}}{\Delta_{m n}}-\frac{h}{D} \frac{\alpha_{n}}{\Delta_{m n}^{2}} \sigma_{13 m}+\frac{p_{0} S}{D} \frac{\sin \alpha_{n} e}{\Delta_{m n}^{2}}\left(\sin \alpha_{m} g+\sin \alpha_{m} q\right)
\end{align*}
$$

Using the boundary condition for the edge $x_{1}=0$ we can calculate the "moment stress" $\sigma_{13}\left(x_{2}\right)$ as

$$
\begin{gather*}
\sigma_{13 m}=\frac{\sum_{n=1}^{\infty} \frac{\alpha_{t}(1+v) \tau_{1 n m} \alpha_{n}}{\Delta_{m n}}+\frac{p_{0} S}{D} \frac{\alpha_{n} \sin \alpha_{n} e}{\Delta_{m n}^{2}}\left(\sin \alpha_{m} g+\sin \alpha_{m} q\right)}{\frac{h}{D} \sum_{n=1}^{\infty} \frac{\alpha_{n}^{2}}{\Delta_{m n}^{2}}} \\
\sigma_{13}\left(x_{2}\right)=\frac{2}{b} \sum_{m=1,3, \ldots}^{\infty} \sigma_{13 m} \sin \alpha_{m} x_{2} \tag{4.6}
\end{gather*}
$$

The solution is consisted of two parts: the influence of the rate of temperature across the plate thickness and the influence of the mechanical forces.

### 4.2 Deformation in the middle surface

Vibrations and stress in the middle surface of the plate can be obtained in analytical form by using differential equation (1.4) and appropriate boundary conditions. In this paper it is done for presented numerical example by using finite element method, Program package KOMIPS [1] author T. Maneski. Input file was done on the analytical solution for the temperature in the middle surface of the plate (3.4).

### 4.3 Numerical example

For numerical values presented in section 3 of this paper, the solutions for the deformation and stress fields are presented on figures 4.2 to 4.5 . Total deformation is presented on figure 4.2. Maximal calculated deformation is 0.281 mm .

Figures 4.3, 4.4 and 4.5 show appropriate stress field. Maximal temperature in the plate for the stationary state is $\theta=10.3^{\circ} \mathrm{C}$ and maximal stress value in the total case of loading, calculated by finite element method, is $1.354 \mathrm{kN} / \mathrm{cm}^{2}$.

Table 4.1 Maximal deformation and maximal stress

|  | Loading |  |  |
| :--- | :---: | :---: | :---: |
|  | Temperature | Mechanical | Total |
| Maximal deformation $f_{\max }[\mathrm{mm}]$ | 0.014 | 0.280 | 0.281 |
| Maximal stress $\sigma_{\max }\left[\mathrm{kN} / \mathrm{cm}^{2}\right]$ | 1.242 | 0.693 | 1.354 |

For the presented numerical example temperature gradient across the plate thickness can be neglected. Analytical results for the transversal vibrations agree with the numerical results.


Fig. 4.2. Total deformation


Fig. 4.4. Mechanical loading. Stress field $\sigma=0.35 \div 0.65 / 0.05 \mathrm{kN} / \mathrm{cm}^{2}$


Fig. 4.3. Total Stress field, $\sigma=0.7 \div 1.3 / 0.1 \mathrm{kN} / \mathrm{cm}^{2}$


Fig. 4.5. Temperature loading. Stress field $\sigma=0.3 \div 1.2 / 0.1 \mathrm{kN} / \mathrm{cm}^{2}$

## 5. Conclusion

Magneto-thermoelasticity has received considerable attention because of the possible applications in detection of flaws in ferrous metals, optical acoustics, levitation by superconductors, magnetic fusion and many other electro-mechanical devices.

The problem of the influence of electromagnet with constant flux on the behaviour of thin metallic plate can be described in analytical form throw four systems of differential equation. In that case, the influence on the stress field has two factors: increasing the temperature (appearance of volume heat source in one part of the plate) and mechanical forces (Maxwell stress) placed in air gaps surfaces.

Very suitable method for solving temperature field and transversal vibrations in analytical form, as it was shown in the paper, is integral transform technique. But for membrane case of loading, dynamic and geometrical complicated problems with non homogeneous boundary conditions it is very difficult to find vibrations and stress in analytical form. So, finite element method has been involved in calculation.

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