

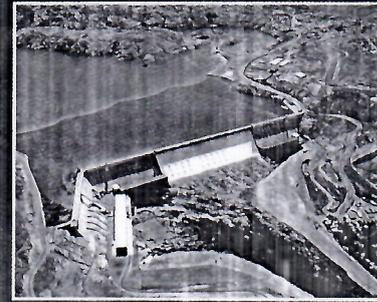
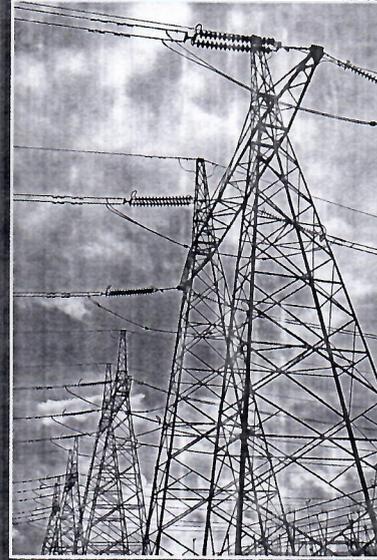
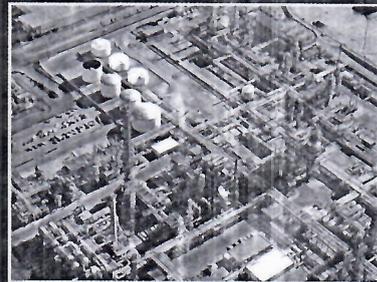
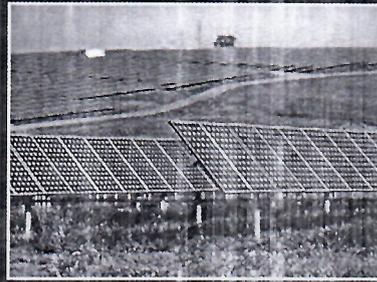
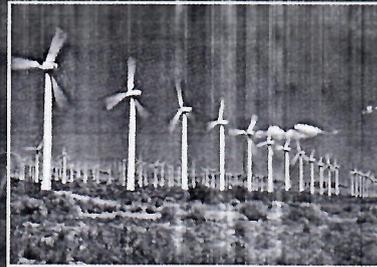
ISSN br. 0354-8651



List Saveza energetičara  
Broj 1-2 / Godina XVII / Mart 2015.  
UDC 620.9

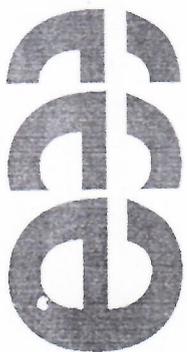
# energija

■ ekonomija ■ ekologija



# ENERGETIKA 2015

# energija



ekonomija ■ ekologija

Energija/Ekonomija/Ekologija

Broj 1-2, mart 2015

Osnivač i izdavač  
Savez energetičara

Predsednik SE  
Prof. dr Nikola Rajaković

Sekretar SE  
Nada Negovanović

Glavni i odgovorni urednik  
Prof. dr Nenad Đajić

Adresa redakcije  
Savez energetičara  
11000 Beograd  
Dečanska 5  
tel. 011/322-6007

E-mail: savezenergeticara@EUnet.rs  
www.savezenergeticara.org.rs

Kompjuterski prelom EKOMARK  
Dragoslav Ješić

Štampa  
„Akademska izdanja“,  
Beograd

Godišnja pretplata  
- 10.000,00 dinara  
- za inostranstvo 20.000,00  
dinara

Tekući račun SE  
broj 355-1006850-61

Radovi su recenzirani uz  
tehničku obradu.  
Nijedan deo ove publikacije  
ne može biti reprodukovan,  
presnimavan ili prenošen bez  
prethodne saglasnosti Izdavača.

## IZDAVAČKI SAVET

*Aleksandar Antić*,  
ministar rudarstva i energetike  
*dr Srđan Verbić*, ministar  
prosvete, nauke i tehnološkog  
razvoja

*Željko Sertić*, ministar privrede  
*Prof. dr Snežana  
Bogosavljević - Bošković*,  
ministar poljoprivrede i zaštite  
životne sredine

*Prof. dr Branko Kovačević*,  
dekan ETF

*Prof. dr Aleksandar Gajić*,  
Mašinski fakultet Beograd

*Prof. dr Slobodan Stupar*,  
pomoćnik ministra

*Prof. dr Zoran Rajić*, državni  
sekretar

*Dušan Mrakić*, ministarstvo  
rudarstva i energetike

*Ljubo Mačić*, dir. Agencije za  
energetiku Srbije

*Dragan Jovanović*, izv. dir.  
EPS

*dr Kiril Kravčenko*, gen. dir.  
NIS ad

*Aleksandar Obradović*,  
direktor JP EPS

*Aleksej Belov*, dir. Bloka  
„Energetika“ NIS

*Čedomir Ponočko*, dir.  
TENT, d.o.o.

*Nikola Petrović*, gen. dir.  
JP EMS

*dr Aca Marković*, JP EPS  
*Dušan Bajatović*, dir.  
JP Srbijagas

*Milorad Grčić*, dir.  
RB Kolubara d.o.o.

*Goran Knežević*, dir. HE  
Đerdap, d.o.o.

*Slobodanka Krčevinac*,  
dir. EDB

*Goran Horvat*, dir. TE-KO  
Kolubara

*mr Bogdan Laban*, dir.  
Elektrovojvodina, d.o.o.

*Tomislav Basta*, v.d. dir. JP  
Transnafta

*Srđan Đurović*, dir.  
Elektrosrbija, d.o.o.

*Aleksandar Vlačić*, dir.  
Obnovljivi izvori EPS

*dr Miroslav Malobabić*, dir.  
JP Srbijagas

*Darko Bulatović*, dir.  
„Jugoistok“ d.o.o.

*Sanja Tucaković*, dir.  
„Centar“, d.o.o.

*Dobrosav Arsović*, dir.  
JKP Novosadska toplana

*Zoran Ivančević*, dir.  
Panonske TE-TO

*Vuk Hamović*, EFT Group  
*dr Nenad Popović*,  
ABS Holding

*dr Dragan Kovačević*, dir.  
Elektrotehnički institut  
„Nikola Tesla“

*Prof. dr Sanja Vraneš*, dir.  
Instituta „Mihajlo Pupin“

*Borislav Grubor*, Instituta za  
nuklearne nauke „Vinča“

*Prof. dr Milorad Milovančević*,  
dekan Mašinskog fakulteta  
u Beogradu

*Prof. dr Dejan Filipović*, dekan  
Geografskog fakulteta

*Prof. dr Šćepan Miljanić*, dekan  
Fakulteta za fizičku hemiju

*Prof. dr Rade Dobroslovački*,  
dekan Fakulteta tehničkih  
nauka u NS

*Prof. dr Ivan Obradović*, dekan  
Rudarsko-geološkog fakulteta  
u Beogradu

*Prof. dr Jeroslav Živanić*,  
dekan Tehnički fakultet u  
Čačku

*Prof. dr Milun Babić*, Fakultet  
inženjerskih nauka u  
Kragujevcu

*Dejan Popović*, N.O. EPS  
*Slobodan Babić*, Rudnap  
Group

*Dr Vladimir Živanović*, SE

## REDAKcioni ODBOR

*Prof. dr Ozren Očić*  
*Slobodan Petrović*, sekretar  
Odbora za energetiku PKS

*Radša Kostić*, dir.  
Elektroistok-izgradnja

*dr Tomislav Simović*, dir.  
Montinvest ad

*Milorad Marković*, predsednik  
HK Minei

*Milan Lončarević*, NIS a. d.

*Prof. dr Petar Đukić*, TMF  
*Dragan Nedeljković*, novinar

*Dr Branislava Lepotić*, dir.  
JP Transnafta

*Jelena Vujović*, dir. za odnose  
s javnošću EPS

*Roman Mulić*, SE

*Simo Bobić*, PK Beograda  
*Nikola Petrović*, dir. Energetika  
Kragujevac

*Ružica Vranjković*, novinar  
*Jelica Putniković*, novinar

# energija

■ ekonomija ■ ekologija

## ORGANIZACIONO - PROGRAMSKI ODBOR

**Predsednik:** Prof.dr Milun Babić, Mašinski fakultet u Kragujevcu

**Sekretar:** Nada Negovanović, sekretar Saveza energetičara

### Članovi:

Prof.dr Беляков Алексей Васильевич, Научно-исследовательский институт» ОАО «ВТИ») – Российская Федерация

Dr Matthias Jochem Mitsubishi Hitachi Power System Europe GmbH, Nemačka

Dr Jean Rizzon, Mitsubishi Hitachi Power System Europe GmbH

Dr Patrick Weckes, Mitsubishi Hitachi Power System Europe GmbH

Prof. dr Miloš Nedeljković, Mašinski fakultet Beograd

Prof. dr Adriana Sida Manea, Politehnica-Universitety of Temisoara, Romania

dr Ivan Souček, Ph. D., Prague Institute of Chemical Technology, Czech Republic

Prof.dr Zoran Rajić, državni sekretar

Prof.dr Slobodan Stupar, pomoćnik ministra

Prof. dr Miloš Banjac, pomoćnik ministra

Prof.dr Branko Kovačević, dekan ETF u Beograd

Prof.dr Aleksandar Gajić, Mašinski fakultet Beograd

Prof.dr Dečan Ivanović, Mašinski fakultet Podgorica

Prof.dr Zdravko N.Milovanović, Mašinski fakultet Banja Luka

Prof.dr Valentino Stojkovski, Mašinski fakultet Skopje

Prof.dr Predrag Popovski, Mašinski fakultet Skopje

Prof.dr Aleksandar Nospal, Mašinski fakultet Skopje

dr Igor Krčmar, Elektrotehnički fakultet Banja Luka

Prof.dr Rade Biočanin, Univerzitet Aperiion Banja Luka

dr Tatjana Luppova, Rusija

dr. D. Seibt, Vattenfall - Nemačka

Prof.dr Nikolaj Ostrovski, Ukrajina

Mihail Cvetkov, Silovije mašini, Rusija

Prof. Daniela Marasova, CSc. Technical university of Kosice

Faculty of Mining, Ecology

Prof dr Dejan Filipović, dekan Geografskog fakulteta

Prof dr Jeroslav Živanić, dekan Tehničkog fakulteta u Čačku

Prof dr Slobodan Vukosavić, Elektrotehnički fakultet Beograd

Prof.dr Milan Medarević, dekan Šumarskog fakulteta u Beogradu

Dr Radoslav Raković, Energoprojekt-Entel a.d.

Prof.dr Mirko Komatina, Mašinski fakultet u Beogradu

Ljubo Mačić, Predsednik Agencije za energetiku Srbije

Prof. dr Gordana Dražić, dekan Fakulteta za primenjenu ekologiju - Futura

Prof.dr Ozren Očić, Faculty of International Engineering Management

dr Tomislav Simović, član UO SE

Dr Miodrag Arsić, IMS Beograd

Prof dr Željko Despotović, IMP

dr Miroslav N.Malobabić, izvršni direktor JP Srbijagas

Prof dr Nenad Đajić, glavni i odgovorni urednik časopisa ENERGIJA

Prof.dr Vladimir Živanović, Savez energetičara

# ess energija

■ ekonomija ■ ekologija

- [254] I. Ristić, B. Radojčić, I. Jovanović, Z. Đurđević, M. Radivojević, P. Tatomirović  
**Modernizacija merno – upravljačkog sistema pomoćnog kotlovskeg postrojenja u TENT B**
- [262] I. Ožegović, Z. Mićević  
**Pripremni radovi za izvođenje hemijskog čišćenja i parnog prođuvavanja kotla**
- [273] P. Tatomirović, N. Turnić, N. Novaković, D. Filimonović, A. Latinović  
**Predlog unapređenja regulacije broja obrtaja turbine napojne pumpe**
- [280] D. Andrašić  
**Uticađ centralne i lokalne regulacije u sistemima daljinskog grejanja na poboljšanju enerđetskih performansi zgrada**
- [289] I. Božić, M. Tanasijević, I. Milović, R. Mitrović, L. Jekić-Aničić, G. Bajić, S. Đorđević  
**Održavanje hidroenerđetskih postrojenja prema riziku od otkaza vitalnih delova hidrauličnih turbina**
- [297] R. Jovanović, I. Božić  
**Primena veštačkih neuronskih mreža u određivanju enerđetskih karakteristika propelernih hidrauličnih turbina**
- [305] V. Stojkovski, A. Nospal  
**Transient Fluid Flow Into Parallel Pipelines Constructed of Pipes with Different Materials**
- [311] Đ. Novković, N. Maričić, M. Jevtić  
**Numerička simulacija strujanja u turbini FRANCIS-99**
- [319] M. Arsić, V. Grabulov, Z. Savić, S. Bošnjak, B. Međo  
**Integrity of Beam Braces and Threaded Spindle for Conjoint Operation of Two 5 MN Bridge Cranes**
- [325] D. Glišić, V. Nešić, G. Konečni  
**Unapređenje postojećeg softverskeg alata za konfigurisanje Atlas Max-RTL po IEC 61850 protokolu**
- [331] S. Polić  
**Dispozicije korišćenja energije vode, umetnost i filozofija prostora**
- [336] J. R. Jelić, I. R. Radović, M. Lj. Kijevčanin, O. J. Ocić  
**Analiza mogućnosti povećanja enerđetske efikasnosti rada postrojenja za fluidni katalitički kreking u sekundarnoj preradi nafte**
- [342] I. Souček, O. Ocić, Z. Popović  
**Dilemma of Refining Industry on Selection of Bottom of the Barrel Technology**
- [348] M. Nikolić Simić, M. Dubajić  
**Obezbeđenje poverenja u kvalitet rezultata ispitivanja sastava prirodnog gasa gasnim hromatografima "Daniel" koji se nalaze na gasovodnom sistemu JP Srbijagas-a**
- [353] M. Imerovic, M. Bucko, M. Vuruna, N. Ivankovic, J. Bogdanov  
**Metanol i dimetil etar kao savremena sintetička goriva**
- [359] V. Veljković  
**Odstupanje pri merenju toplotne vrednosti procesnim gasnim hromatografom na sistemu «Srbijagas»-a u zavisnosti od atmosferskeg pritiska - SRPS ISO 17025**

Miodrag Arsić, Vencislav Grabulev, Zoran Savić

Institute for materials testing, Belgrade, Serbia

Srđan Bošnjak

Faculty of Mechanical Engineering, Belgrade, Serbia

Bojan Međo

Faculty of Technology and Metallurgy, Belgrade, Serbia

UDC: 621.873.004

# Integrity of Beam Braces and Threaded Spindle for Conjoint Operation of Two 5 MN Bridge Cranes

## ABSTRACT

The beam connects two bridge cranes with the overall lifting capacity of 500 t (2x250 t) and enables their simultaneous conjoint operation during the refurbishment or capital repair of hydroelectric equipment at the hydro power plant "Djerdap 2". Two braces are being installed instead of two hooks when that situation occurs. Braces are loaded with 250 t (2.5 MN) each, while threaded spindle is loaded with 500 t (5 MN).

Integrity of structures is a relatively new scientific and engineering discipline which in a broader sense comprises state analysis, behaviour and loosening diagnostics, service life evaluation and refurbishment of structures, which means that, beside the usual situation in which it is necessary to evaluate the integrity of a structure when a flaw gets detected by means of non-destructive tests, this discipline also comprises structural stress state analysis.

Detection of internal defects by means of ultrasonic testing was performed in order to analyze the state of braces and of the threaded spindle. On the basis of performed analytical calculations it was determined that their integrity would not be jeopardized during the refurbishment or capital repair of hydroelectric equipment at the hydro power plant "Djerdap 2".

**Key words:** cranes, non-destructive testing, braces, threaded spindle, structural integrity

## 1. INTRODUCTION

Non-destructive testing performed in order to determine the current state of components of the beam for simultaneous conjoint operation of two bridge cranes with overall lifting capacity of 500 t (5 MN) is only a preparatory action required for their operation during refurbishment or capital repair of equipment at HPP 'Djerdap 2'. On the basis of test results the overall lifting capacity and integrity are being checked. The beam connects two bridge cranes and

enables their simultaneous conjoint operation, figure 1 (marked 1 at figure 1b). Threaded spindle (marked 2 at figure 1b) is loaded with 500 t, while braces (marked 3 at figure 1b), which are being installed instead of hooks during the simultaneous conjoint operation of cranes, are loaded with 250 t each. According to design documentation [1], braces and threaded spindle were made of steel with guaranteed chemical composition OLC 35 (romanian label) by forging. Chemical composition is presented in table 1, while mechanical properties are given in tables 2 and 3.

Table 1. - Chemical composition, values in [%]

Steel	C	Si	Mn	Cr	Ni	Mo	S	P
OLC 35	0.32-0.39	max 0.4	0.5 - 0.8	max 0.4	max 0.4	max 0.1	max 0.045	max 0.045

Table 2. - Mechanical properties of steel OLC 35 for the thickness of brace forging  $t = 220$  mm

Steel	Yield strength, YS [N/mm <sup>2</sup> ]	Tensile strength, TS [N/mm <sup>2</sup> ]	Elongation, A5 [%]
OLC 35	245	500	min 15

Table 3. - Mechanical properties of steel OLC 35 for the thickness of threaded spindle forging  $t = 900$  mm

Steel	Yield strength, YS [N/mm <sup>2</sup> ]	Tensile strength, TS [N/mm <sup>2</sup> ]	Elongation, A5 [%]
OLC 35	210	470	min 15

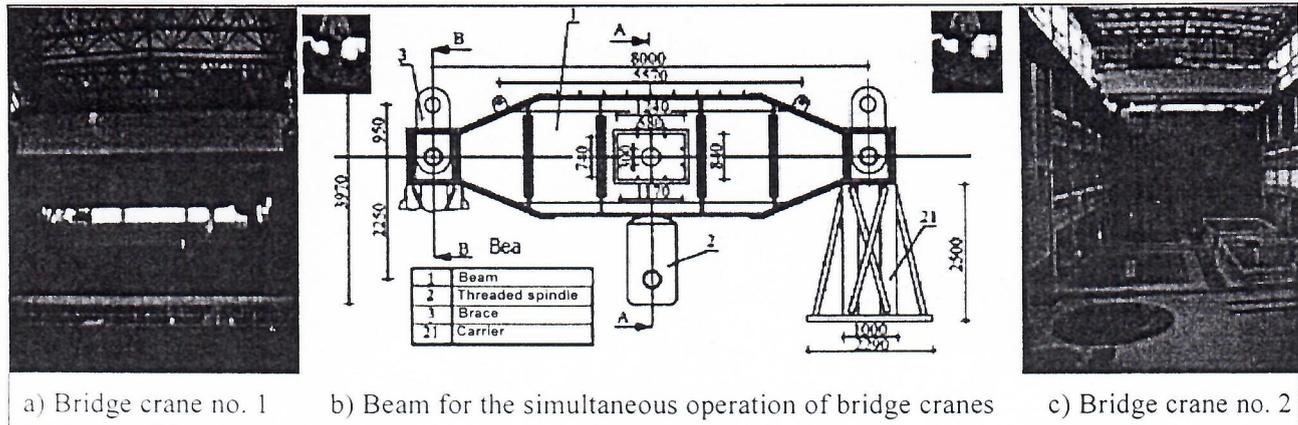


Figure 1. - Appearance of the beam for simultaneous conjoint operation of two bridge cranes

## 2. NON-DESTRUCTIVE TESTING OF BEAM BRACES AND THREADED SPINDLE

In order to perform analysis of the current state, check the lifting capacity and evaluate the integrity of vital beam components (braces, threaded spindle), the following was carried out:

- visual testing (VT) of all components of beam equipment,
- magnetic particle testing (MT) in order to detect surface defects,
- ultrasonic testing (UT) of internal homogeneity,
- metallographic tests by replica method in order to determine the structure of material,
- hardness testing.

### 2.1 Visual Testing of All Components of Beam Equipment

Visual testing of all components of beam equipment confirmed the existence of products of corrosion, as well as the existence of insignificant mechanical damages. Mechanical damages were repaired by fine

grinding, and afterwards all components were submitted to sandblasting and application of anti-corrosive protection.

### 2.2 Magnetic Particle Testing Performed at Braces and Threaded Spindle

No surface defects were detected during the magnetic particle testing of braces and threaded spindle for the simultaneous conjoint operation of bridge cranes.

### 2.3 Ultrasonic Testing of Internal Homogeneity of Braces and the Threaded Spindle

In figures 2 and 3 models and ultrasonic test results for the right brace of the beam (the only one at which internal inhomogeneities were detected) and threaded spindle are shown. Findings, marked with red and yellow colour, refer to areas at which the testing was carried out. On the basis of results of ultrasonic testing, shown in figures 2 and 3, it can be concluded that detected findings are nothing but impurities of lamellar shape grouped around the central area that originated during the forging process, which is a relatively co-

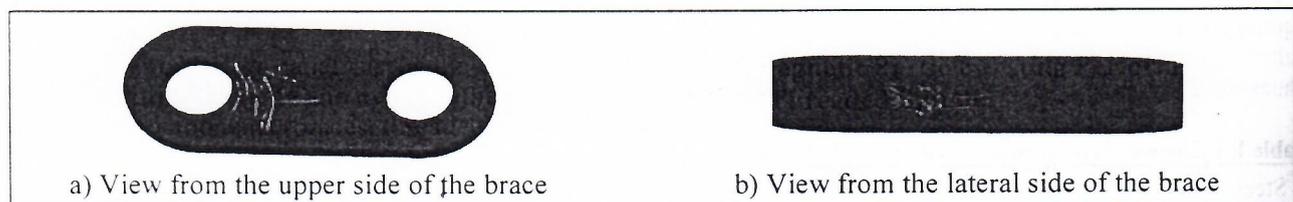


Figure 2. - Results of ultrasonic testing performed at the right brace

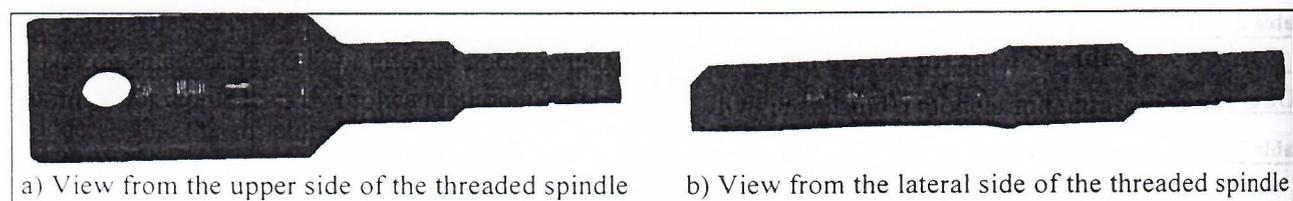


Figure 3. - Results of ultrasonic testing performed at the threaded spindle

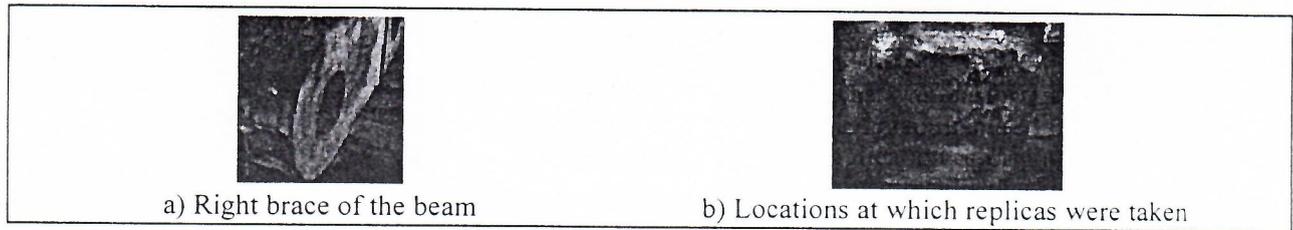


Figure 4. - Locations at which replicas R1A and R2A were taken, right brace of the beam

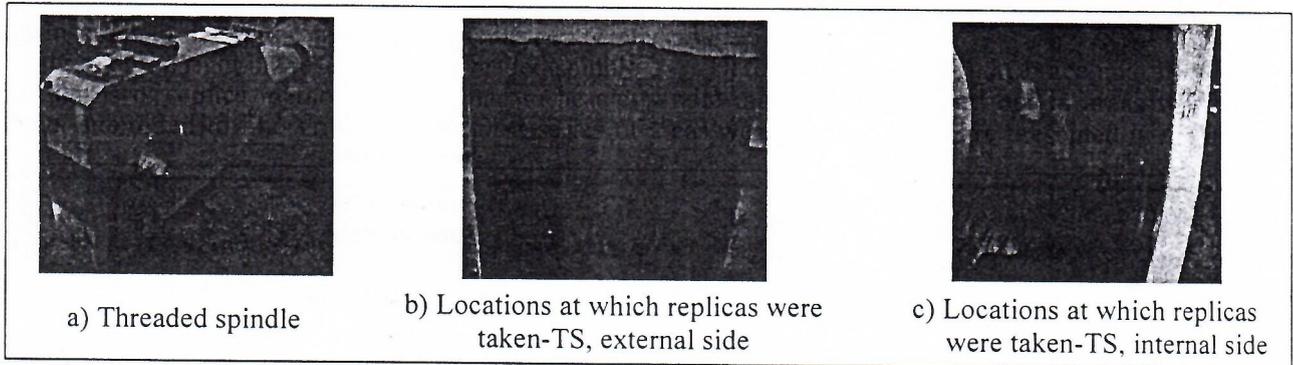


Figure 5. - Locations at which replicas (R1, R2, R3, R4, R5, R1B and R2B) were taken, threaded spindle

common occurrence during the production of large forgings.

#### 2.4 Metallographic Testing of Right Brace and Threaded Spindle by Means of Replica Method

Metallographic testing of right brace (B) and threaded spindle (TS) material was performed by means of replica method [2]. Locations at which replicas were taken are shown in figures 4 and 5.

In order to determine the microstructural state of material of the right brace and of the threaded spindle by means of replica method surfaces had to be cleaned, submitted to fine grinding through a series of

appropriate operations, polished, rinsed and, finally, etched by means of 4% nital. Metallographic analysis of taken replicas was performed by light microscope 'METAVAL', manufactured by 'Carl Zeiss', Jena, through the application of brightfield technique. A selection of obtained results that refer to the right brace is presented in figure 6, while results that refer to the threaded spindle are presented in figure 7.

#### 2.5 Hardness Testing

Hardness testing on right brace and threaded spindle material was carried out in areas where replicas were taken. Results are presented in table 4. The

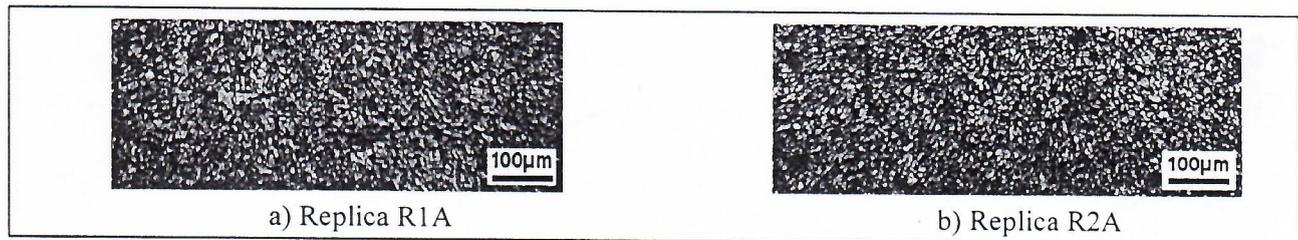


Figure 6. - Fine-grained ferrite-pearlite microstructure with of corrosion products (B)

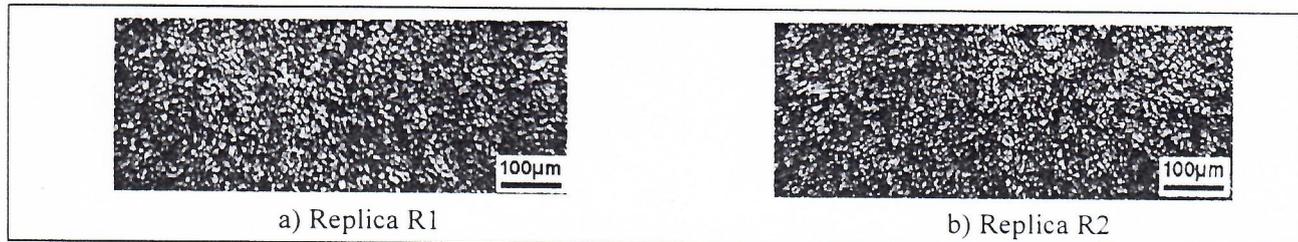


Figure 7. - Fine-grained ferrite-pearlite microstructure with of corrosion products (TS)

Table 3. - Mechanical properties of steel OLC 35 for the thickness of threaded spindle forging  $t = 900 \text{ mm}$

Measurement locations	R1	R2	R3	R4	R5	R1B	R2B	R1A	R2A
Values in HB	135-137	133-135	139-141	140-143	137-141	138-142	138-139	157-164	165-167

overall conclusion is that measured hardness values respond to prescribed values for used materials. The difference in values obtained at the brace with respect to values obtained at the threaded spindle could be explained by smaller thickness of the brace forging. Table 4. - Hardness test results in HB

### 3. ANALYTICAL CALCULATION OF BRACES AND THREADED SPINDLE

The analytical calculation was performed in order to check the lifting capacity and evaluate the integrity of braces and threaded spindle during the simultaneous operation of two bridge cranes.

#### 3.1 Analytical Calculation of Beam Brace Stress

According to manufacturers' documentation, braces were made of steel OLC 35 [1]. Taking into account the fact that the calculation was carried out for the allowable stress  $S_{all} = 120 \text{ Mpa}$ , it can be concluded that the safety factor with respect to yield strength (table 2) is:

$$S_{BM} = \frac{YS_{0.2}}{S_{all}} = \frac{245}{120} = 2 \quad (1)$$

Dimensions and critical cross-sections used for the analytical calculation of beam braces are shown in figure 8.

##### 3.1.1 Calculation of specific stress caused by pressure at the opening of the brace

Specific stress caused by 5 MN pressure at the opening of the brace:

$$S_{PB} = \frac{Q}{2 \cdot d_o \cdot s} = \frac{5000000}{2 \cdot 300 \cdot 220} = 37,9 \text{ MPa} \leq S_{all} \quad (2)$$

Safety factor for the calculated stress  $S_{all}$ :

$$S_B = \frac{120}{37,9} = 3,17 \quad (3)$$

Overall safety factor:

$$S_O = 2,0 \times 3,17 = 6,34 \quad (4)$$

##### 3.1.2 Calculation of stress at the internal surface of the brace opening, cross-section A1-B1

Stress at the internal surface of the brace opening for 5 MN pressure, cross-section A1-B1:

$$S_{A1-B1} = \frac{S_{pBi} \left[ (2R)^2 + d^2 \right]}{(2R)^2 - d^2} = \frac{37,9 \left[ (2 \cdot 445)^2 + 280^2 \right]}{(2 \cdot 445)^2 - 280^2} = 59,2 \text{ MPa} \leq S_{all} \quad (5)$$

Safety factor for the calculated stress:

$$S_{BM} = \frac{120}{59,2} = 2,03 \quad (6)$$

Overall safety factor:

$$S_O = 2,0 \times 2,03 = 4,06 \quad (7)$$

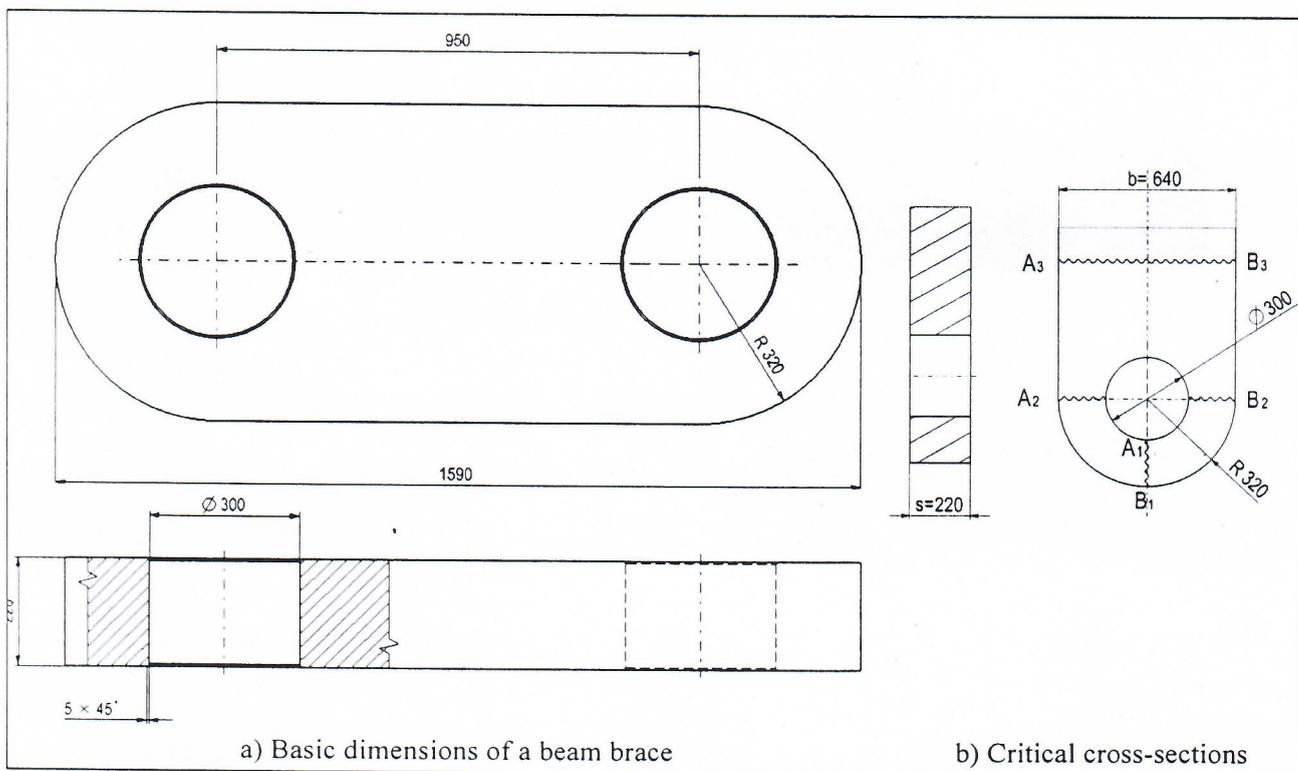


Figure 8. - Dimensions and critical cross-sections used for the calculation of beam braces

**3.1.3 Calculation of stress at the internal surface of the brace opening, cross-section A2-B2**

Stress at the internal surface of the brace opening for 5 MN pressure, cross-section A2-B2:

$$S_{A2-B2} = \frac{Q}{2 \cdot (2R - d) \cdot s} = \frac{5000000}{2 \cdot (2 \cdot 320 - 300) \cdot 220} = 33,4 \text{ MPa} \leq S_{all} \quad (8)$$

Safety factor for the calculated stress:

$$S_{BM} = \frac{120}{33,4} = 3,59 \quad (9)$$

Overall safety factor:

$$S_O = 2,0 \times 3,59 = 7,18 \quad (10)$$

**3.2 Analytical Calculation of Stress at the Threaded Spindle**

Dimensions and critical cross-sections used for the analytical calculation are shown in figure 9.

**3.2.1 Calculation of stress at the cross-section with diameter 298mm of the threaded spindle**

Stress at the cross-section with diameter 298 mm of the threaded spindle loaded with  $Q_{TS} = 5 \text{ MN}$ :

$$S_{TS} = \frac{5000000 \cdot 4}{3,14 \cdot 298^2} = 71,7 \text{ MPa} \leq S_{all} \quad (11)$$

Safety factor for the calculated stress:

$$S_{TS} = \frac{120}{71,7} = 1,67 \quad (12)$$

Overall safety factor:

$$S_O = 1,75 \times 1,67 = 2,92 \quad (13)$$

**3.2.2 Calculation of specific stress caused by pressure at the opening of the eye of the threaded spindle**

Specific stress caused by pressure at the opening of the eye of the TS for load  $Q_{TS} = 5 \text{ MN}$ :

$$S_{pTS} = \frac{Q}{d_o \cdot s} = \frac{5000000}{280 \cdot 304} = 58,7 \text{ MPa} \leq S_{all} \quad (14)$$

Safety factor for the calculated stress:

$$S_{TS} = \frac{120}{58,7} = 2,04 \quad (15)$$

Overall safety factor:

$$S_O = 1,75 \times 2,04 = 3,57 \quad (16)$$

**3.2.3 Calculation of stress at the internal surface of the threaded spindle eye opening, cross-section A1-B1**

Stress at the internal surface of the eye opening of the threaded spindle for  $Q_{TS} = 5 \text{ MN}$ , cross-section A1-B1:

$$S_{A1-B1} = \frac{S_{pTSr} \cdot [(2R)^2 + d^2]}{(2R)^2 - d^2} = \frac{58,7 \cdot [(2 \cdot 445)^2 + 280^2]}{(2 \cdot 445)^2 - 280^2} = 71,6 \text{ MPa} \leq S_{all} \quad (17)$$

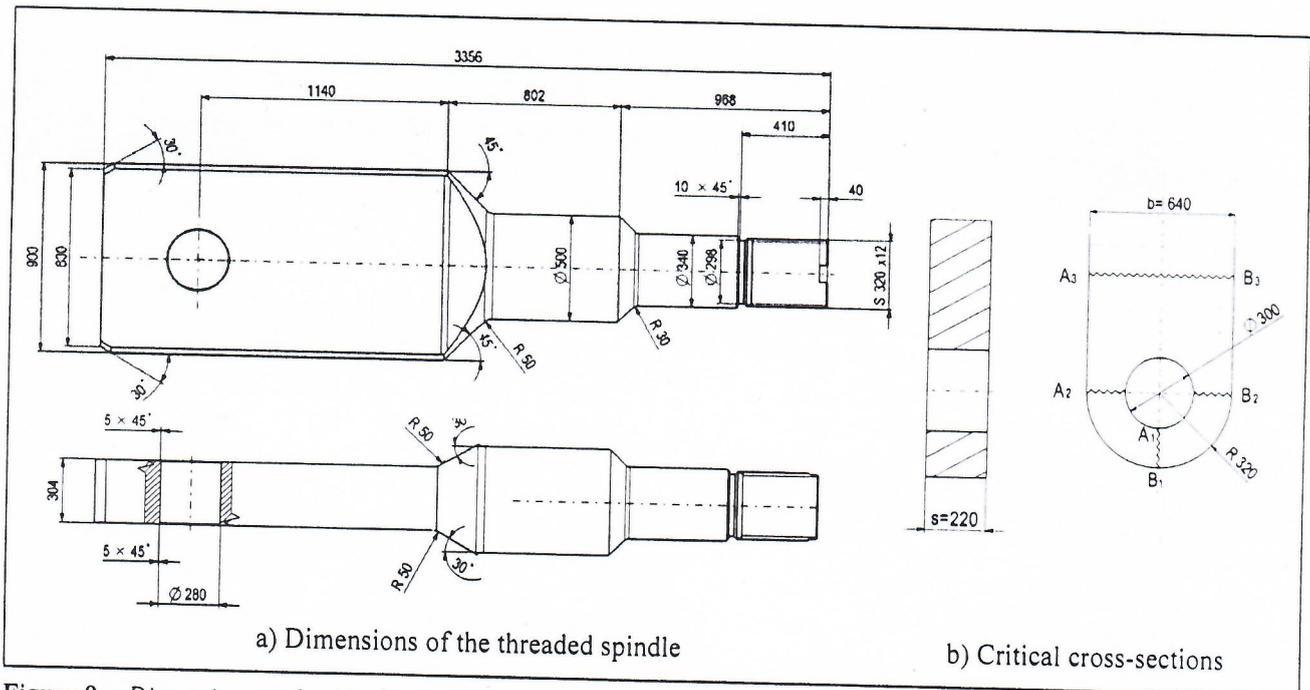


Figure 9. - Dimensions and critical cross-sections required for the calculation of the threaded spindle

Safety factor for the calculated stress:

$$S_{BM} = \frac{120}{33,4} = 3,59 \quad (18)$$

Overall safety factor:

$$S_O = 1,75 \times 1,68 = 2,94 \quad (19)$$

**3.2.4 Calculation of stress at the internal surface of the eye opening of the threaded spindle, cross-section A2-B2**

Stress at the internal surface of the eye opening of the threaded spindle for  $Q_{TS} = 5 \text{ MN}$ , cross-section A2-B2:

$$S_{A2-B2} = \frac{Q}{(2R-d) \cdot s} = \frac{5000000}{(2 \cdot 445 - 280) \cdot 304} = 27,0 \text{ MPa} \leq S_{all} \quad (20)$$

Safety factor for the calculated stress:

$$S_{TS} = \frac{120}{27} = 4,44 \quad (21)$$

Overall safety factor:

$$S_O = 1,75 \times 4,44 = 7,77 \quad (22)$$

**4. CONCLUSION**

It can be concluded that non-allowable internal non-homogeneities detected by means of ultrasonic testing of right brace and threaded spindle do not affect the lifting capacity of components required for the simultaneous conjoint operation of two 5 MN bridge cranes, because the analytical stress calculations, presented by appropriate safety factors, showed that practically one third of thickness in characteristic cross-sections of right brace and threaded spindle enable the required integrity of vital beam components during the refurbishment or general repair at HPP 'Djerdap 2'.

**ACKNOWLEDGEMENT**

The authors acknowledge the support from the Serbian Ministry of Education, Science and Technological Development for financing Projects TR 35002 and TR 35006.

**LITERATURE**

- [1] Design documentation: „CCSITMRTU”, Timișoara, Romania.
- [2] Arsić M., Veljović A.: State analysis and evaluation of lifting capacity of beam equipment components for conjoint operation of bridge cranes, Institute for Materials Testing, Belgrade, 2013.