PREDICTION OF TAPPING FORCES AND TORQUE FOR 16MnCr5 ALLOYED STEEL

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Preliminary note

The paper presents cutting tools geometry analysis for the active part of straight fluted taps and a prediction model for tapping forces and torque. To derive the relation between the angles that define the geometry of the tools, the orientation matrices for taps coordinate systems are defined. Along with analytical model setup, the taps are additionally modelled in CAD package based on main parameters taken from the external database. The cutting coefficients for material combination for the workpiece (16MnCr5) and tool (HSS-E, EMo5Co5) were identified using the orthogonal cutting mechanics which is needed for prediction procedure. Proposed prediction model for tapping forces and torque of the machine tap, in addition to the effects of the major cutting edges, includes an action from minor cutting edges, too. Results of experimental verification for the proposed procedures and models are presented in this paper.

Keywords: CAD; cutting forces; prediction; tool geometry; tapping

Predviđanje sila i momenta urezivanja navoja u legiranom čeliku 16MnCr5

Prethodno priopćenje

U radu je prikazana analiza geometrije aktivnog dijela rezne oštrice ureznika sa ravnim žljebovima i model za predviđanje sila i momenta urezivanja navoja. Da bi se dobio odnos između kutova koji definiraju geometriju alata, određene su orijentacijske matrice za koordinatne sustave ureznika. Pored postavljanja analitičkog modela, ureznici su modelirani u CAD paketu s glavnim parametrima iz vanjske baze podataka. Koeficijenti rezanja za kombinaciju materijala obratka (16MnCr5) i alata (HSS-E, EMo5Co5) identificirani su pomoću mehanike ortogonalnog rezanja. Predloženi model za predviđanje sila i momenta urezivanja strojnim ureznikom, osim učinka glavnih reznih oštrica, uključuje također i akcije pomoćnih oštrica. Prikazani su rezultati eksperimentalne provjere predloženih postupaka, kao i usporedba s predviđenim vrijednostima sila i momenta.

Ključne riječi: CAD; geometrija alata; predviđanje; sile rezanja; urezivanje navoja

1 Introduction

Metal cutting is considered as one of the most important methods for material removing in the production of mechanical components. The prediction of cutting forces acting on the workpiece is essential for solving several important issues: estimation of the power of a machine tool; estimation of the straining actions that must be resisted by the machine tool components, jigs and fixtures; evaluation of the significance of various parameters of cutting forces; evaluation of the performance of new workpiece materials, tool materials, etc. with respect to machinability [1].

All cutting operations share the same principles of cutting mechanics, but their geometry and kinematics are different. The first step in the prediction of forces acting on a cutting tool is to consider a relatively simple orthogonal cutting process in which the cutting edge is perpendicular to the cutting speed and the deformation occurring in the plane. The results of this analysis are used as a base for the development of a model for a much more general case of oblique cutting, where the edge is angled to the cutting speed [2].

This paper is focused to the prediction of tapping forces and torque. Tapping is a common operation used to produce internal threads in the predrilled hole by means of special tool, named the tap. This is one of the more demanding machining processes. A tap is a cutting tool with very complex geometry.

The reference survey shows that the number of papers on tapping issues is rather small, and consequently, the number of models proposed for specific cutting geometry problems is weak as well.

For several decades, it has remained troublesome and costly due to various process failures such as tap

breakage, oversized and undersized threads, and burrs at the threaded holes entry and exit [3].

Armarego [4] presented predictive modelling of various technological performance measures for the wide spectrum of machining operations used in practice. He has shown that the cutting process for the straight flute taps could be represented as a series of elemental single edge oblique cuts on each active cutting edge at the tap chamfer end.

Cao and Sutherland [5] developed a mechanistic model for tapping torque and axial force prediction. It is shown that the total tapping load consists of a load resulting from the chip formation and tool flank friction, and a chip packing load, random in nature, resulting from chip clogging in the flute.

Dogra et al. [6] analyzed a mechanistic torque and thrust model for the tapping process. The basic concept involved in that model is that the elemental cutting and thrust forces acting on each tooth are proportional to the uncut chip area. The proportionality constant is determined for a given tool workpiece combination and in a given cutting range by fitting experimental data through drilling experiments used for calibration.

Armarego and Chen [7] and Chen and Smith [8] studied computer-aided predictive models for force and torque in machine tapping operations, based on Armarego's "Unified-Generalised Mechanics of Cutting Approach" for optimizing the generation and use of taps. The results confirm the validity and generic nature of the unified-mechanics of cutting approach.

In order to avoid tap breakage and low threads quality, it is important to obtain a reliable model for force and torque prediction. In their studies, many authors considered only the geometry and cutting action of the major tap cutting edges (Fig. 1, detail "A"). This paper introduces the influence of tap minor edges as additional effect to the major ones. The content is structured as follows:

Section 2 of this paper describes the procedure to identify straight fluted tap geometry at arbitrary point on the cutting edge, tap minor edges and chamfer angle influences.

Section 3 describes the process of tap geometry formation in CAD and MATLAB program which is used to identify position and orientation of each elementary edge.

Section 4 includes the force and torque prediction procedure that considers major and minor edges influence, and prediction results. This procedure is illustrated by an example of prediction for tapping 16MnCr5 alloyed steel workpiece material with M10 straight fluted machine tap. Further, experimental results and comparison with the predicted values are shown.

2 Straight fluted taps geometry

The tool-in-hand and the tool-in-use reference system of planes can be defined for any tool. The tool-in-hand geometry includes a set of geometric elements, which are defined through the tool drawing, used in manufacturing, sharpening or measurement of the tool. The tool-in-use geometry works with real or effective geometric elements of cutting tools, which appear in the cutting operations [9].

Fig.1 shows the tool-in-hand and the tool-in-use geometry of the cutting tool part of machine tap with three straight flutes, with the nominal diameter D, the tool back rake γ_p and the tool cutting edge angle κ_r , at arbitrary selected point 0 on the cutting edge.



Figure 1 Tool-in-hand and tool-in-use geometry of machine tap

Projection of the cutting edge is shown in the plane P_r which is normal to cutting speed v at selected point. Assumed working plane P_f contains the cutting speed and feed speed. The cutting edge is approximately a straight line due to the small length of cut and rake angle. The cutting edge is inclined at approximately the chamfer angle κ_{r_p} to the plane P_f as shown in the plane P_r .

Conventions for the description and presentation of the coordinate system and the indexing system of the transformations are from the theory of computer graphics and modelling systems of solids, and are also used in robot manipulation problems [10].

An arbitrary point in the space P, can be presented with the position vectors with respect to a variety of coordinate systems, so for coordinate systems in which the origin coincides [10]:

$${}^{1}p = {}^{1}_{2}R \cdot {}^{2}p$$
, then ${}^{2}p = {}^{1}_{2}R^{-1} \cdot {}^{1}p = {}^{1}_{2}R^{T} \cdot {}^{1}p$, (1)

because the inverse of an orientation matrix, R, is the same as its transposed matrix.

Based on Fig. 1, the switch from the tool-in-hand coordinate system "f" defined with the basic planes P_r , P_f and P_p , labelled as "1", into the tool-in-use system "4" defined with planes P_n , P_{se} and their normal plane that contains intersection $P_{re} \cap P_n$, requires three orientation matrices:

$${}_{2}^{1}\boldsymbol{R} = \boldsymbol{R}_{y,\kappa_{\mathrm{T}}} = \begin{bmatrix} \cos\kappa_{\mathrm{T}} & 0 & \sin\kappa_{\mathrm{T}} \\ 0 & 1 & 0 \\ -\sin\kappa_{\mathrm{T}} & 0 & \cos\kappa_{\mathrm{T}} \end{bmatrix},$$

$${}_{3}^{2}\boldsymbol{R} = \boldsymbol{R}_{x,\lambda_{\mathrm{S}}} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos\lambda_{\mathrm{S}} & -\sin\lambda_{\mathrm{S}} \\ 0 & \sin\lambda_{\mathrm{S}} & \cos\lambda_{\mathrm{S}} \end{bmatrix},$$

$${}_{4}^{3}\boldsymbol{R} = \boldsymbol{R}_{z,\psi_{\mathrm{T}}} = \begin{bmatrix} \cos\psi_{\mathrm{n}} & -\sin\psi_{\mathrm{n}} & 0 \\ \sin\psi_{\mathrm{n}} & \cos\psi_{\mathrm{n}} & 0 \\ 0 & 0 & 1 \end{bmatrix},$$
(2)

where κ_r denotes the chamfer angle, λ_s is the angle of inclination and ψ_n is the angle between the cutting edge planes in the tool-in-hand (P_s) and tool-in-use (P_{se}) geometry, measured in the normal plane P_n. In the coordinate system "4", oblique cutting is identified using the results obtained by the orthogonal cutting experiments.

Coordinates of the end point of the resultant cutting velocity vector v_e in the first (plane P_f) and the last coordinate system (plane P_{se}) according to Fig. 1, are known:

$${}^{1}\boldsymbol{p} = -v_{e} \begin{bmatrix} 0 & \cos\eta & \sin\eta \end{bmatrix}^{\mathrm{T}},$$

$${}^{4}\boldsymbol{p} = -v_{e} \begin{bmatrix} 0 & \cos i & \sin i \end{bmatrix}^{\mathrm{T}},$$
(3)

where η represents the resultant cutting speed angle and *i* is the oblique angle. From the expression:

$${}^{1}\boldsymbol{p} = {}^{1}_{2}\boldsymbol{R} \cdot {}^{3}_{3}\boldsymbol{R} \cdot {}^{4}_{4}\boldsymbol{R} \cdot {}^{4}\boldsymbol{p} , \qquad (4)$$

unknown angles are derived:

$$\tan \psi_{\rm n} = \frac{\tan \eta \sin \kappa_{\rm r}}{\cos \lambda_{\rm s} + \tan \eta \cos \kappa_{\rm r} \sin \lambda_{\rm s}} \,. \tag{5}$$

The rake angle, γ_{n0} , in the normal plane P_n , at selected point 0 on the cutting edge is defined, according to Fig. 1, from the expression:

$$\tan \gamma_{n0} = \tan \gamma_{p0} \frac{\cos \lambda_s}{\cos \kappa_r} - \tan \kappa_r \sin \lambda_s \,. \tag{6}$$

Accordingly, the tool-in-use rake angle, γ_{ne0} , in the normal plane P_n , at selected point 0 on edge is:

$$\gamma_{\rm ne0} = \gamma_{\rm n0} + \psi_{\rm n} \,. \tag{7}$$

The angle η is defined in the tool-in-use coordinate system as the feed speed over tangential cutting speed:

$$\tan \eta = \frac{v_{\rm s}}{v} = \frac{P}{\pi d_0} \,. \tag{8}$$

The relation between angles of the tool-in-hand geometry is:

$$\tan \lambda_{\rm s0} = \tan \gamma_{\rm p0} \sin \kappa_{\rm r0} \,, \tag{9}$$

and angle γ_{p0} as a function of γ_{p} :

$$\gamma_{\rm p0} = \arcsin\left(\frac{D}{d_0}\sin\gamma_{\rm p}\right),\tag{10}$$

where D, d_0 represent tool diameters.

The oblique angle, i, in the tool-in-use cutting edge plane P_{se} at selected point 0 on the cutting edge is defined as follows:

$$\tan i = \frac{\sin \eta \cos \kappa_{\rm r} \cos \lambda_{\rm s} - \cos \eta \sin \lambda_{\rm s}}{\sin \eta \sin \kappa_{\rm r}} \sin \psi_{\rm n} \,. \tag{11}$$

2.1 Influence of minor edges and chamfer angle

Most often, prediction of tapping forces includes only the action of major edges, but the influence of minor edges is not negligible, especially on the torque. The influence on the axial force is considerably low, assuming symmetry of minor edges relative to the axial plane. The length ratio of minor edges and major edge maximum, for standard ISO profile with the thread angle $\alpha = 30^\circ$, can be expressed by the expression:

$$\frac{l_{\rm p}}{b_{\rm g\,max}} = \frac{\cos\left(30 - \kappa_{\rm r}\right)}{\tan\kappa_{\rm r}\left(3\cdot\cot^2\kappa_{\rm r} - 1\right)} \cdot \frac{10,67}{z},\qquad(12)$$

where l_p is the sum of minor edges lengths on a single tooth, $b_{g max}$ is the maximal major edge length and z is the number of flutes. The influence of the chamfer angle κ_r on this ratio, for the number of flutes z = 3, 4 and 6 is depicted in Fig. 2.



Figure 2 The influence of angle κ_r on a length ratio of minor and major edges

It is evident form Fig. 2 that the influence of minor edges length increases with the chamfer angle κ_r . Therefore, at higher κ_r values, the length of minor edges can be expected to be higher relative to that of the major edges. This urges the need to study the influence of minor edges in the cutting process and their inclusion in the force model.

3 Machine taps modelling

For the straight fluted taps, the major and minor edges can be analytically obtained at cross-sectional surfaces forming them, or by using CAD software packages. In this paper, a class of taps is modelled in the Autodesk Inventor 2011CAD programming environment, using the technique of parametric modelling. In this way, the input into the CAD software is a set of parameters that describe specific dimensions of the taps. One CAD model is created based on tap technology, and works with a number of different sets.

The parameters can be saved in .xml format or spreadsheet software program format. This model uses Microsoft Excel spreadsheet format.xls, which can be defined in similar open-source programs (e.g., Apache OpenOffice). The program for preparing a table with the parameters is based on the nominal diameter, tap pitch, and tap type as inputs.

Fig.3 shows the image of cutting edges of the used machine tap M10 magnified 35 times, and simultaneously displays the image of M10 tap model, obtained by the CAD software using a set of parameters.



Figure 3 The cutting edges of the M10 tap in one flute $(\times 35)$ and the CAD model

Fig. 4 displays the cutting edges along the flutes of the M10 tap with the chamfer angle κ_r =18°, which were obtained in the program package MATLAB using the tap geometry from the CAD package.



4 Cutting forces and torque in tapping

Many studies have shown that the influence of the uncut chip thickness h and chip width b on the cutting force is more significant than the depth of cut a_p and feed f [4÷8]. For example, in turning, the cross-section of cutting, with the same depth of cut and the feed has a different form, depending on the tool cutting edge angle. Linear force model between the cutting forces and a product of the uncut chip thickness and chip width is commonly used.

The tap is cut into thin slices (dz) perpendicular to the axial direction [11÷13]. The discretized tap, thin disc, and the directions of cutting forces in tapping are given in Fig. 5.



Figure 5 Discretized cutting length of the tap (a), Disc of differential thickness (b), Differential cutting forces in oblique cutting on differential length of the cutting edge (c)

Based on the modified mechanics of cutting analysis [2, 4], the three elemental forces dF_t , dF_r and dF_a consist of a "cutting" force component, and an "edge" force component due to rubbing or ploughing at the cutting edge. It is convenient to express the cutting forces in the following form [2]:

$${}^{4}dF_{t} = K_{tc}bh + K_{te}b,$$

$${}^{4}dF_{r} = K_{rc}bh + K_{re}b,$$

$${}^{4}dF_{a} = K_{ac}bh + K_{ae}b,$$
(13)

where ${}^{4}dF_{t}$ is the cutting force or main force acting in the direction of the cutting velocity, ${}^{4}dF_{r}$ is the radial force acting perpendicularly to the machined surface and ${}^{4}dF_{a}$ is the axial feed force acting in the direction of the tool travel in the coordinate system "4".

The corresponding cutting force constants are [2]:

$$K_{\rm tc} = \tau_{\rm s} / C \cdot \left(\cos \left(\rho - \gamma_{\rm n} \right) + \tan i \, \tan \nu \sin \rho \right),$$

$$K_{\rm rc} = \tau_{\rm s} / C \cdot \left(\cos \left(\rho - \gamma_{\rm n} \right) \tan i - \tan \nu \sin \rho \right),$$

$$K_{\rm ac} = \tau_{\rm s} / C \cdot \cos i \cdot \left(\sin \left(\rho - \gamma_{\rm n} \right) \right),$$
(14)

where τ_s is the shear stress, ρ represents the average friction angle, v is the chip flow angle and K_{te} , K_{re} , K_{ae} are edge force coefficients in tangential, radial and axial directions. The parameter *C* is given as follows:

$$C = \sin \phi_n \sqrt{\cos^2(\phi_n + \rho - \gamma_n) + \tan^2 \nu \sin^2 \rho} , \qquad (15)$$

where ϕ_n represents shear angle.

Oblique cutting performs the major edge inclined at an angle κ_r , and two minor edges inclined at an angle α and $-\alpha$, where α is the standard thread angle. The major cutting edges can be found on the conical chamfer surface, while the minor edges are located on the flanks of the threads. Fig. 4 shows the cutting edges on the conical chamfer surface obtained from the machine tap CAD model to determine the cutting edges length and position. Now, applying

$${}^{1}dF = \left[dF_{\rm t} \ dF_{\rm r} \ dF_{\rm a} \right]^{\rm T} = {}^{1}_{2} \mathbf{R} \cdot {}^{2}_{3} \mathbf{R} \cdot {}^{4}_{4} {\rm R} \cdot {}^{4} dF , \qquad (16)$$

the cutting force, F_z , and torque, M, can be evaluated from:

$$F_z = \sum_i {}^1 dF_{a_i}, \quad M = \sum_i {}^1 dF_{t_i} \cdot r_i .$$
 (17)

If we are also interested in calculations in the machine fixed coordinate system, namely forces in two perpendicular directions F_x and F_y and side force F_s , respectively, it is needed, according to Fig. 5c, to perform the transformation into this system as follows:

$${}^{0}\boldsymbol{p} = {}^{0}\boldsymbol{p}_{0} + {}^{0}\boldsymbol{R} \cdot {}^{1}\boldsymbol{p} , \qquad (18)$$

where ${}^{0}\boldsymbol{p}_{0}$ is the position vector of the origin and ${}^{0}_{1}\boldsymbol{R}$ is the orientation matrix of fixed coordinate system relative to the moving coordinate system.

4.1 Identification of cutting coefficients

Cutting coefficients can be found from a database, which usually contains values for limited number of combinations of tool and workpiece materials [7, 8]. For material combinations not supported by a database, simple cutting experiments can provide necessary data [2, $4, 12\div14$].

The cutting coefficients can be modelled using either orthogonal cutting mechanics or mechanistic models. In this paper the cutting coefficients were identified using the orthogonal cutting mechanics which was used in predicting the cutting forces and torque. Different methods for determining the influence on cutting surface [15] and the edge forces are discussed by Gonzalo et al. [16] and Popov et al. [17], such as the direct method, the comparison method and the extrapolation method on the zero uncut chip thickness, based on which the forces on the front face do not influence the edge forces.

The cutting process was modelled as a function of work material, tool material and geometry, chip load and cutting speed, by conducting orthogonal cutting tests on the lathe. Experiments are done with cutting fluids switched off. Workpiece was a tube with a 2,1 mm wall thickness.

In order to obtain cutting coefficients necessary for analysis presented in this work, facing tools featuring diverse rake angles $(5\div20^\circ)$ were manufactured using same HSS-E steel (EMo5Co5) as that used for machine taps serial production. The tools had a sharp cutting edge.

Spindle speeds of $5\div 20$ m/min and feed rates of $0,01\div 0,15$ mm/rev were used. Also, the rake angle is selected within the range of $5\div 20$ degrees. Feed in orthogonal cutting represents the uncut chip thickness, *h*.

Fig. 6 presents the machining and acquisition systems applied to determine the forces in orthogonal cutting. Cutting forces, main force F_1 and feed force F_2 , were measured in two directions using a custom made, two-component, strain gauge force dynamometer, amplifiers HBM KWS 3082A and NI CompactDAQ acquisition platform (NI USB 9174, NI 9215).

Collection of chip samples was also done in order to calculate its thickness h_c and chip ratio r_c . The mean cutting forces from steady-state phase of experiments for constant cutting speed and rake angle are calculated and fitted as a linear function of uncut chip thickness. Fig. 7 shows one of force vs. uncut chip thickness diagram from

orthogonal cutting test results, for cutting speed v = 10 m/min and rake angle $y = 20^{\circ}$.



Figure 6 Experimental setup for orthogonal cutting

Parameters ρ and r_c , were identified using statistical analysis of data provided from orthogonal cutting tests, while τ_s was simply averaged for the same series of experimental data. The average edge coefficients K_{te} and K_{re} were obtained by means of linear regression and extrapolation of measured forces to zero uncut chip thickness and represent the edge forces per unit width [2, 14]. These results obtained for the material combination of workpiece (16MnCr5) and tool (HSS-E, EMo5Co5) are presented in Tab. 1.



Figure 7 Cutting forces during orthogonal turning of 16MnCr5 tubes $(\nu=10\text{m/min},\nu=20^\circ)$: measurement and linear regression

Fable	1 Ort	hogonal	cutting	data	based	on ort	hogona	l cutting	tests
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$\tau_{\rm s} = 558 \text{ MPa}$					
$\rho = 35,18 + 0,627\gamma$					
$r_{\rm c} = C_0 h C_1$					
$C_0 = 0,942 - 0,012\gamma$					
$C_1 = 0,391 - 0,01\gamma$					
$K_{\rm te} = 28,8 {\rm N/mm}$					
$K_{\rm ae} = 21,5 {\rm N/mm}$					

4.2 Prediction results

Tapping forces and torque were predicted using presented equations based on cutting edge geometry along the flutes and experimentally obtained cutting coefficients. The model is performed in program package MATLAB. Predicted, through hole tapping axial force and torque for the M10 C type tap (tap pitch P = 1,5 mm, chamfer angle $\kappa_r = 18^\circ$, and rake angle $\gamma_p = 12^\circ$) with cutting speed v = 5 m/min for workpiece-tool material combination 16MnCr5/ EMo5Co5 is presented in Fig. 8.



Figure 8 Predicted tapping axial force and torque (M10 tap, d = 8,4 mm, P = 1,5 mm, $\kappa_r = 18^\circ$, $\gamma_p = 12^\circ$, n = 160 rpm, material combination 16MnCr5/ EMo5Co5)

During tapping a through hole with straight flute tap, the force and torque are increased as each edge on a chamfer section of the tap enters the operation (A-B, Fig. 8) and maximum constant value is reached when all edges are involved in cutting operation (B-C). When chamfer section is disengaged with the workpiece (C-), axial force and torque fall to zero.

4.3 Experimental verification of prediction

Experiments were performed on numerically controlled machine tool – horizontal machining centre LOLA HMC500 with no cutting fluids. The experimental setup for tapping is presented in Fig. 9. For the force and torque measurement, a two-component piezoelectric dynamometer Kistler 9271, amplifiers Kistler 5007 and NI CompactDAQ 9174 with module NI 9215 were used.



Figure 9 Experimental setup for acquisition of axial force and torque during the tapping cycle

Measured axial force and torque and predicted values for equal conditions are presented in Fig. 10, for M10 C type tap with metric normal profile. Measurement signal was filtered using cubic smoothing spline to reduce axial cutting force and torque dynamics component. Geometric and process parameters used for experimental model verification are listed in Tab. 2. Taps are with three straight flutes. The workpiece-tool material combination was 16MnCr5/ EMo5Co5. Cutting speed was 5 m/min, and through hole depths were 15 mm.



force and torque during the M10 tapping operation

 Table 2 Geometric and process parameters used for experimental model

 verification

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Description	Unit	Value					
Tap material		HSS-E, EMo5Co5					
Nominal diameter	mm	10					
Pitch	mm	1,5					
Chamfer length	mm	2,9					
Rake angle	deg	12					
Number of flutes	-	3					
Workpiece material		16MnCr5					
Premachined hole diameter	mm	8,4					
Through hole length	mm	15					
Spindle speed	min ⁻¹	160					

Axial force and torque measurement signals slightly increase during constant predicted values, as the influence of tap calibrating section friction effects. Overall, a good agreement is obtained. The deviations between experimental and simulation results are expected, however satisfactory, being within the range of 10 %. It is experimentally proven that this model can be used in predicting the tapping forces and torque. The complexity of real tapping process and resulting cutting forces exceed somewhat the capabilities of proposed numerical model. Observed inaccuracy of the model applied in this study comes predominantly from experimentally provided cutting coefficients used for force calculation. Effects of coolant and cooling process, tapping axis position, deviations of diameter of previously machined holes, tool

holder features and many other factors not taken into consideration affect results of modelling as well.

5 Conclusion

A model for the prediction of tapping forces and torque is developed in this research. In addition to the effects of the major cutting edges, this model includes an action from minor cutting edges of the machine tap. The forces and torque acting on a cutting tool during the process are of fundamental importance in the design of cutting tools and evaluation of the performance of new workpiece and tool materials.

Acceptable results, experimentally confirmed and evaluated, were achieved by the prediction related to taps with straight flutes. For the applied workpiece-tool material combination (16MnCr5 / EMo5Co5), the deviations between the predicted and measured values were within the 10 % range, which indicates that the developed model can be effectively used to predict the forces and torque for the tapping process. Using this model, substantial time and cost savings can be obtained.

The parameters identified for a specific combination of workpiece and tool material and presented in Tab. 1, were used to simulate the process of tapping, can be used to simulate any other cutting process. In such a case, it is only necessary to perform a geometry analysis of the new cutting tool.

6 References

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